



Aalto University
School of Engineering

EEN-E2002 Combustion Technology

Internal Combustion Definitions and Characteristics

January 2019, Martti Larmi

Teaching personnel



Martti Larmi, Professor

Department of Mechanical Engineering
martti.larmi@aalto.fi, room: 126f/K3



Tuomas Paloposki, Senior lecturer

Department of Mechanical Engineering
tuomas.paloposki@aalto.fi, room: 205/K4



Ossi Kaario, Senior Research Fellow

Department of Mechanical Engineering
ossi.kaario@aalto.fi, room: 126/ K3



Mika Järvinen, Associate professor

Department of Mechanical Engineering
mika.jarvinen@aalto.fi, room 306/K4

Cheng Qiang, Post doc

Department of Mechanical Engineering
qiang.cheng@aalto.fi, room 150/K3

Otto Blomstedt, Laboratory manager

Department of Mechanical Engineering
otto.blomstedt@aalto.fi, room: 151/K3

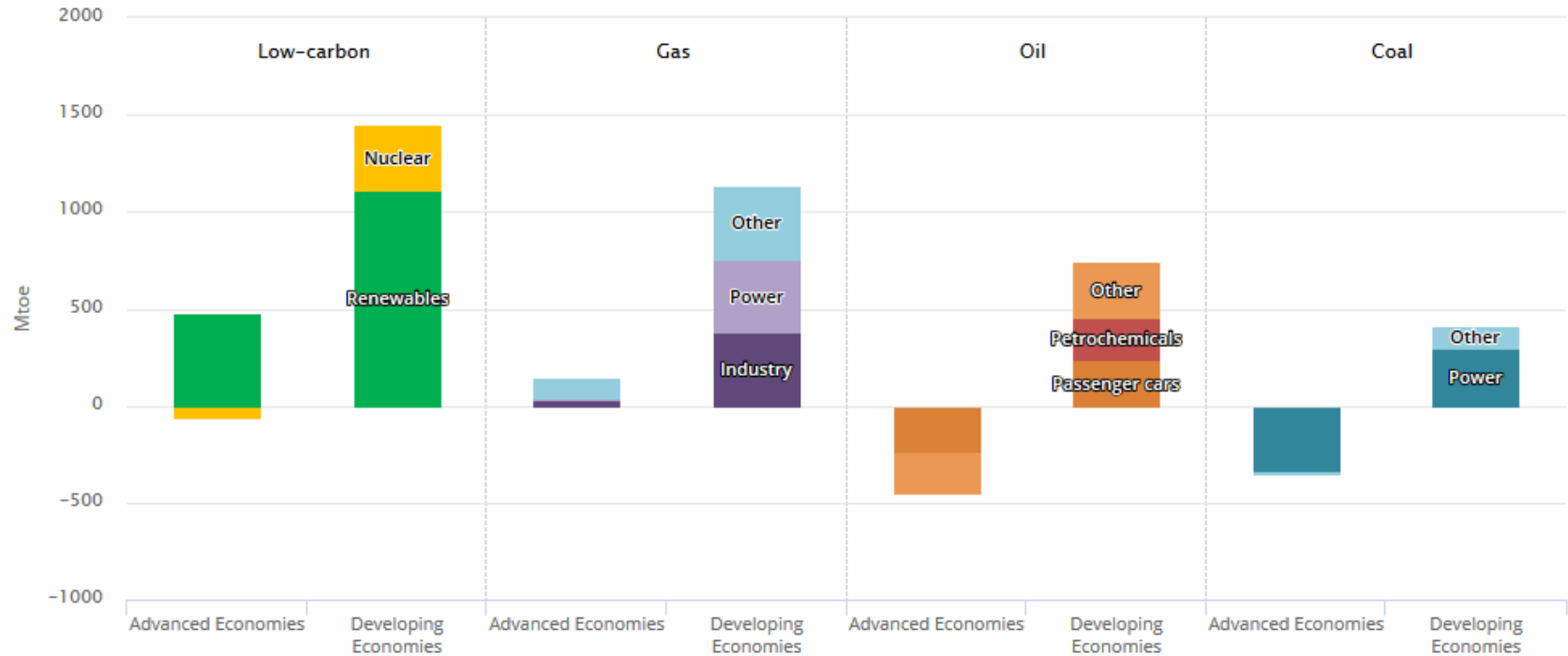
Learning Outcomes / Contents

Learning Outcomes: The student should be able to understand the basics of combustion and gasification processes and to be able to recognize how they influence the design and operation of practical equipment such as boilers and engines.

Contents: Today's combustion technologies and combustion regimes in power plants and engines; design and operational considerations. Application of combustion to furnaces and boilers, spark ignition engines, diesel engines, gas engines and gas turbines. Fundamentals of gasification.

IEA World Energy Outlook 2018

Change in total primary energy demand, 2017-40 in the NPS



© OECD/IEA

Recommended further courses

AAE-E3030 - Numerical Modeling of Multiphase Flows L, periods IV-V

EEN-E3002 - Power Process Simulation, periods IV-V

Passing the course / Prerequisites

Assessment Methods and Criteria:

Six learning exercises (6 LEs) do make 100% of the grade. Alternatively 50% LEs and examination 50%. Each learning exercise is graded and at least 50% of the available points should be taken in each learning exercise. Beside the LEs, there are 2 lab exercises graded pass/ no pass.

Prerequisites: ENY-C2001 Thermodynamics and Heat Transfer or equivalent knowledge and CHEM-A1250 Principles of Chemistry or equivalent knowledge. Recommended pre-studies: EEN-E1030 Thermodynamics in Energy Technology

EEN-E2002 Combustion Technology							
2018-2019		Periods III-IV			Ver 2	January 8, 2019	
Changes still possible							
Week	Date	Time	Event	Location	Topic	Teacher	Exercises
		Sat					
		Sun	Combustion Basics				
2	7.1.2019	Mon					
	8.1.2019	Tue					
	9.1.2019	Wed	10-12	Lecture	K3 118	Engine technology basics	Martti
	10.1.2019	Thu	10-12	Lecture	K3 118	Combustion basics	Ossi
	11.1.2019	Fri	8-10	Learning exercise	K3 118	Learning exercise advice session	Jonny
		12.1.2019	Sat				
		13.1.2019	Sun	Combustion Basics + Disassembly exercise			
3	14.1.2019	Mon					
	15.1.2019	Tue					
	16.1.2019	Wed	10-12	Lecture	K3 118	Engine combustion I	Ossi Separate schedule
	17.1.2019	Thu					
	18.1.2019	Fri	8-10	Learning exercise	K3 118	Learning exercise advice session	Jonny
		19.1.2019	Sat				
		20.1.2019	Sun	Engine Combustion Technology			
4	21.1.2019	Mon					
	22.1.2019	Tue					
	23.1.2019	Wed	10-12	Lecture	K3 118	Engine technology basics	Martti Separate schedule
	24.1.2019	Thu	10-12	Lecture	K3 118	Engine combustion II	Martti
	25.1.2019	Fri	8-10	Learning exercise	K3 118	Learning exercise advice session	Jonny
		26.1.2019	Sat				
		27.1.2019	Sun	Engine Combustion Technology			
5	28.1.2019	Mon					
	29.1.2019	Tue					
	30.1.2019	Wed	10-12	Lecture	K3 118	Fuel admission & air management	Martti Separate schedule
	31.1.2019	Thu	8-18	Excursion to Wärtsilä	Turku training center		Otto
	1.2.2019	Fri					

Textbooks on Internal Combustion

Internal combustion engine handbook : basics, components, systems, and perspectives

edited by Richard van Basshuysen and Fred Schäfer

SAE International 2004

ISBN: 0-7680-1139-6

Additional material, John B. Heywood: Internal Combustion Engine Fundamentals, McGraw-Hill 1988, ISBN:0-07-100499-8

Important Chapters

Ch 2 Definition and Classification of Reciprocating Piston Engines

Ch 3 Characteristics

Ch 10 Charge Cycle

Ch 11 Supercharging of Internal Combustion Engine

Ch 12 Mixture Formation and Related Systems

Ch 13 Ignition

Ch 14 Combustion

Ch 15 Combustion Systems

Basic definitions

Combustion engines convert the chemical energy of fuel to mechanical energy as a result of combustion

Combustion engines can be divided into internal and external combustion engines

External combustion engines

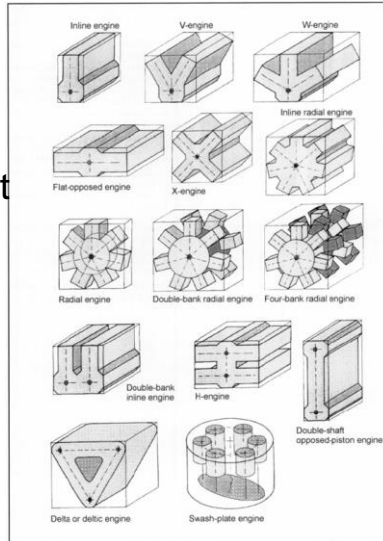
- Stirling engines
- Steam engines

Internal combustion engines

- Reciprocating piston engines
- Rotary piston engines
- Gas turbines

Classification

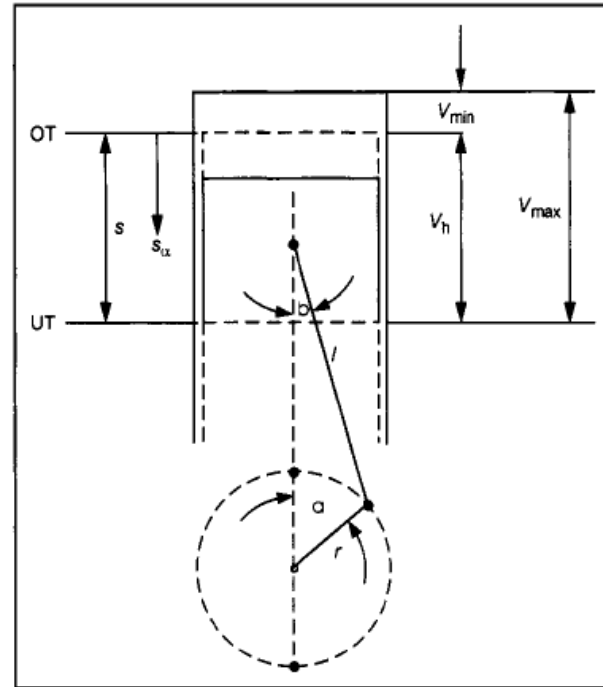
1. Combustion process
2. Fuel
3. Working cycles
4. Mixture generation
5. Gas exchange control
6. Supercharging
7. Configuration
8. Ignition
9. Cooling
10. Load adjustment
11. Application
12. Speed



Type of working process		Open process			Closed process			
		Internal combustion			External combustion			
		Combustion gas = working fluid			Combustion gas ≠ working fluid			
Change of phase of the working fluid								
		No		Yes				
Type of combustion		Cyclical combustion			Continuous combustion			
Type of ignition		Auto-ignition	Supplied ignition					
Machine type		Engine	Diesel	Hybrid	Gasoline	Rohs ⁴	Stirling ⁵	Steam ⁶
		Turbine	—	—	—	Gas	Superheated steam	Steam
Mixture type		Heterogeneous (in the combustion chamber)		Homogeneous (heterogeneous)	Heterogeneous (in the continuous flame)			

Basic structure of reciprocating piston engine

- Cylinder
- Piston
- Piston pin (wrist pin, gudgeon pin)
- Piston rings
- Connecting rod (conrod)
- Crankshaft
- Valves (intake & exhaust)
- Bottom dead center BDC, UT
- top dead center TDC, OT



Basic dimensions

Cylinder bore [m]	D
Piston stroke [m]	S
Cylinder number	z
Stroke-bore ratio	S/D
Crank radius [m]	r
Connecting rod length [m]	l
Connecting rod ratio	$\lambda_s = r/l$
Displacement vol. [m ³]	$V_h = \pi D^2 / 4 * S$ (one cylinder)
Compression volume [m ³]	V_c
Compression ratio	ε (or e) = $(V_h + V_c) / V_c$
Crank angle [deg or rad]	φ or α

Engine size always refers to the total displacement volume of the engine (all cylinders included).

Operational characteristics

Rotational speed [r/min (rpm) or rps]

n

Mean piston speed [m/s]

$c_p = 2Sn (c_m)$

Power [kW]

P_e

Torque [Nm]

M

Brake Mean Effective [bar or Pa]

$p_{e, BMEP}$

Volumetric efficiency

$\lambda_l (\eta_{vol})$

Excess-air factor

$\lambda (\lambda_{tot}, \lambda_c)$

Specific fuel consumption [g/kWh]

b_e

Total efficiency

η_e

Fuel net heating value [kJ/kg]

H_u, H_n

(Lower calorific value)

Excess-air factor, relative air to fuel ratio, equivalence ratio

Excess-air factor λ is the ratio of the air mass in the cylinder to the stoichiometric air mass

$$\lambda = \frac{m_L}{m_{L,St}} = \frac{m_L}{m_K L_{St}} = \frac{L}{L_{St}}$$

m_K is the fuel mass delivered in the cylinder

L_{St} is the requirement of air for stoichiometric combustion [kg/kg], so that the chemical reactions are totally finished. Typically for fuels used in internal combustion engines, such as gasoline and diesel oil, L_{St} is about 14,5 kg/kg

L is the mass of air per the mass of fuel.

The equivalence ratio Φ is the inverse of the λ . Equivalence ratio is widely used in US literature.

Rich combustion: $\lambda < 1$

Lean combustion: $\lambda > 1$

Total lambda λ_{tot} is based on total air flow through the engines

Combustion lambda λ_c is based on the air trapped in the cylinder

$$\Phi = \frac{1}{\lambda}$$

Mean piston speed, compression ratio

Engine type	Max. speed [rpm] approx.	Mean piston speed [m/s] approx.
Racing engine (Formula 1)	18 000	25
Small engines (two-stroke)	20 000	19
Motorcycle engines	13 500	19
Car SI engine	7 500	20
Car diesel engines	5 000	15
Truck diesel engines	4 200	14
Larger high-speed diesel engines	2 200	13
Medium high-speed engines (diesel)	1 200	10
Crosshead engines (two-stroke diesel)	200	8

Engine type	ε		Limited by
	From	To	
Two-stroke SI engine	7.5	10	Autoignition
SI engine (two-valve)	8	10	Knock, autoignition
SI engine (four-valve)	9	11	Knock, autoignition
Direct injection SI engine	11	14	Knock, autoignition
Diesel (indirect injection)	18	24	Loss of efficiency at full load, component load
Diesel (direct injection)	17	21	Loss of efficiency at full load, component load

Power, Torque, Mean Effective Pressure

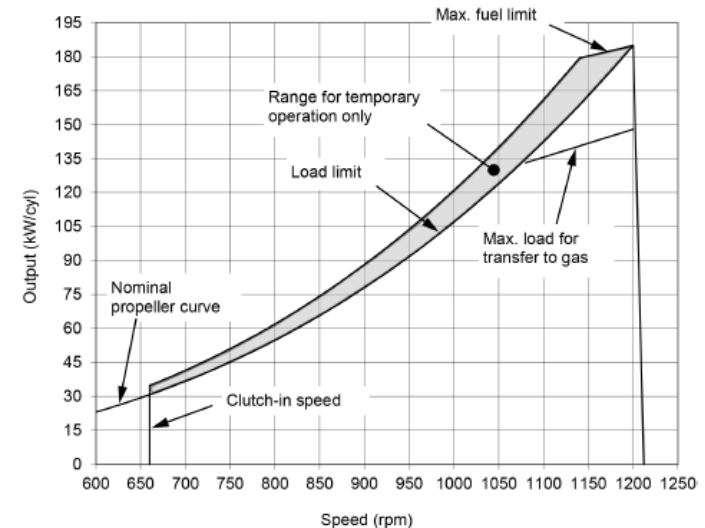
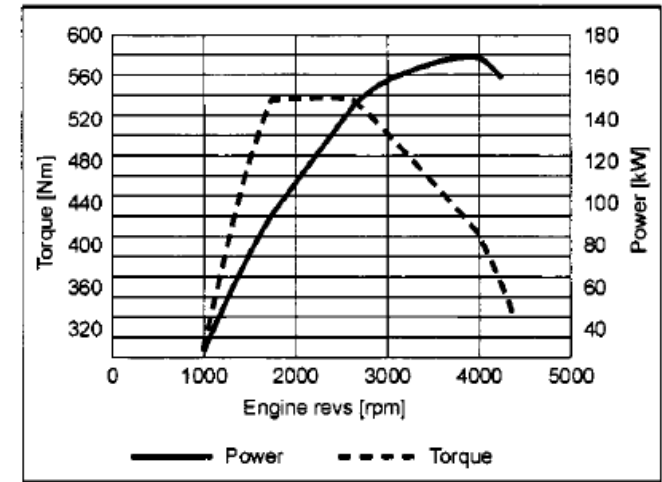
The brake power at any working point of an engine is calculated from the torque and engine rotational speed

$$P_e = M_d \omega = M_d 2\pi n$$

Conclusion: increase in the power can be achieved either by increasing the torque or increasing the engine rotational speed

Brake mean effective mean pressure is a calculated value. It corresponds to a pressure level at which the gases have to work against the piston in order to get the actual work done by the engine or cylinder

$$W = p_e \frac{\pi d_k^2}{4} s$$



Brake mean effective pressure

Brake mean effective pressure describes engine load and the torque that you are able to get out of a certain displacement volume. It is not the average pressure in cylinder!

$$P_e = p_e \frac{\pi d_k^2}{4} S n i$$

Brake power of an engine is the effective mean pressure multiplied by the displacement volume and rotational speed.

i is the number of working cycles per revolution (0,5 for 4-stroke and 1 for 2-stroke engines)

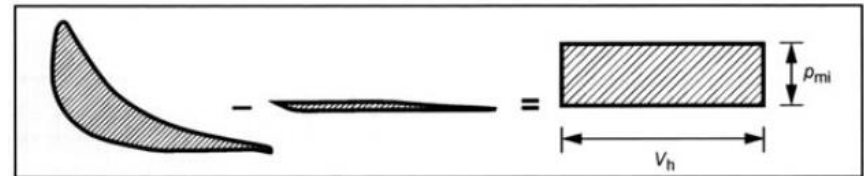
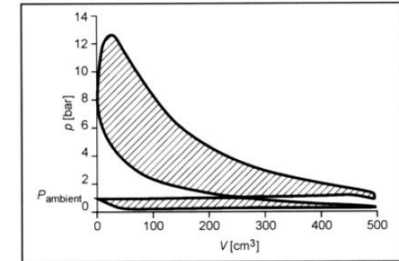
Actual brake power of an engine is also the torque multiplied by the angular velocity. So torque is proportional to the effective mean pressure and the displacement volume

Brake mean effective pressure

Engine type	Effective mean pressure [bar]
	up to
Motorcycle engines	12
Racing engines (Formula 1)	16
Car SI engines (without turbocharger)	13
Car SI engines (with turbocharger)	17
Truck diesel engines (with turbocharger)	22
Car diesel engines (with turbocharger)	20
Larger high-speed diesel engines	30
Medium-speed diesel engines	25
Crosshead engines (two-stroke diesel)	15

$$P_e = p_e \frac{\pi d_k^2}{4} S n i = M_d \omega = M_d 2\pi n$$

$$M_d = p_e \frac{\pi d_k^2}{4} S \frac{1}{2\pi} i$$



p-V-diagram

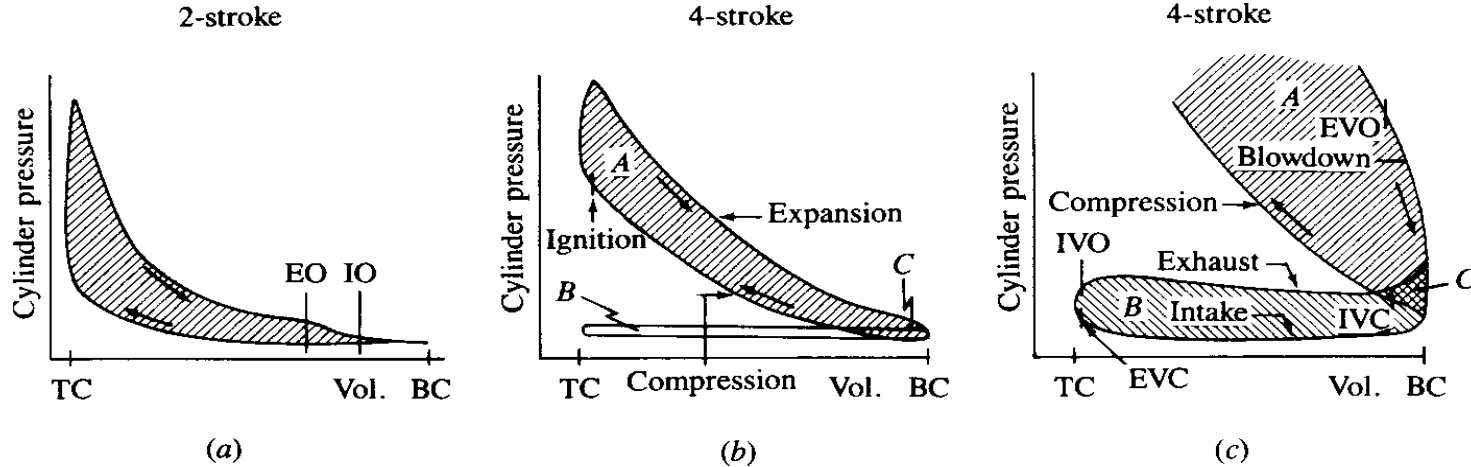
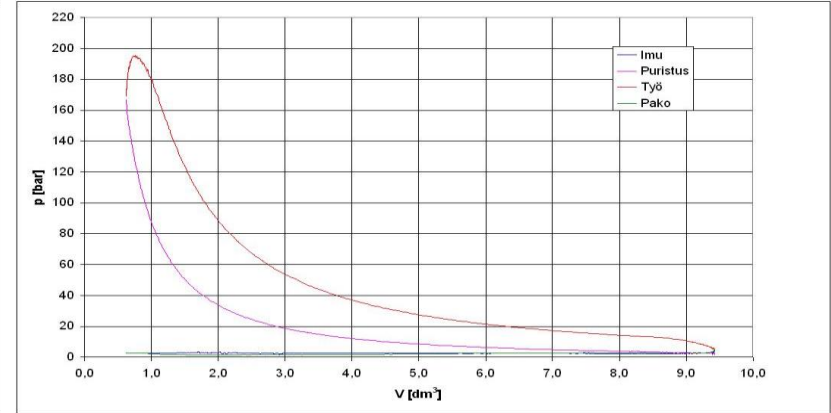
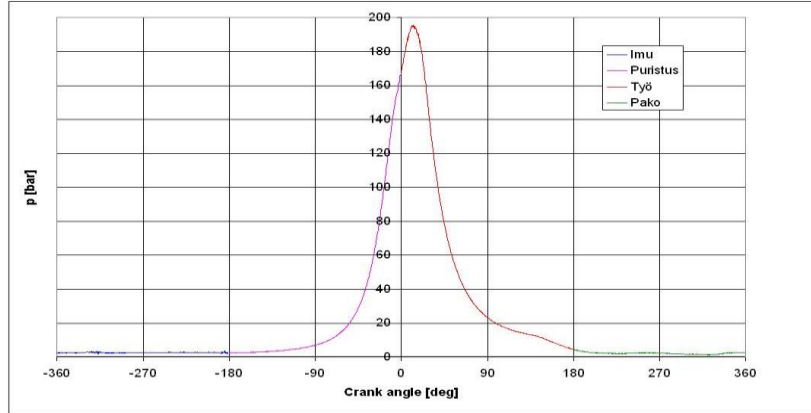


FIGURE 2-4

Examples of p - V diagrams for (a) a two-stroke cycle engine, (b) a four-stroke cycle engine; (c) a four-stroke cycle spark-ignition engine exhaust and intake strokes (pumping loop) at part load.

Cylinder pressure vs. pV-diagram



Wärtsilä 6L20 cylinder pressure and pV-diagram

The real pV-diagram differs from the theoretical diagrams

1. Combustion
2. Heat transfer
3. Exhaust blowdown
4. Gas exchange

IMEP = Indicated mean effective pressure = based on the work done by the gases

FMEP = friction mean effective pressure

BMEP = IMEP-FMEP

IMEP(720) = IMEP gross = Indicated mean effective pressure based on gas work over 720 deg CA, normally IMEP= IMEP(720)

IMEP(360) = IMEP net = Indicated mean effective pressure based on gas work over 360 deg CA

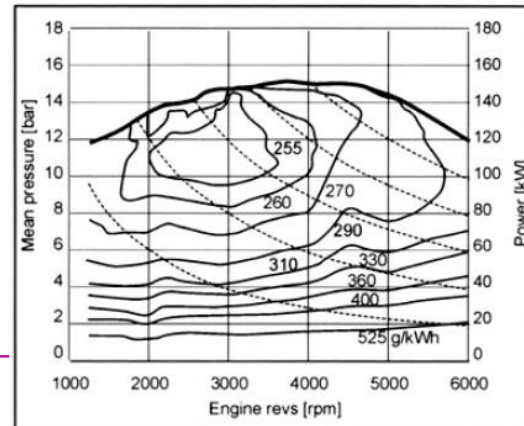
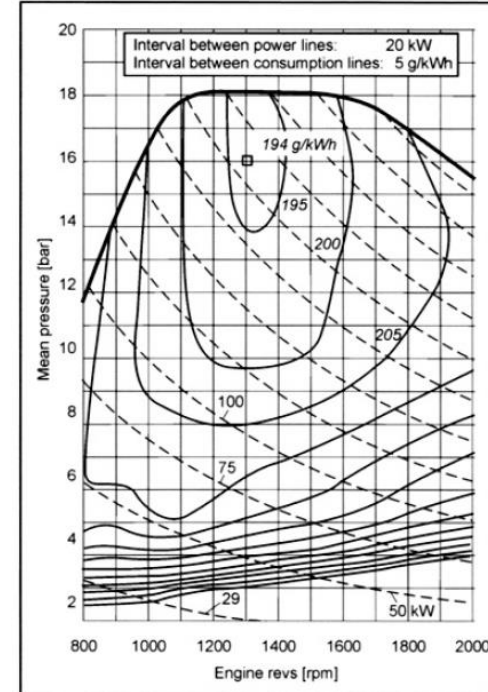
PMEP = pumping mean effective pressure = based on the work done by the gases during gas exchange

IMEP(720) = IMEP(360) + PMEP

More definitions: Heywood Chapter 13.2

Specific fuel consumption

Engine type	Specific fuel consumption [g/kWh] up to	Efficiency [%] up to
Small engines (two-stroke)	350	25
Motorcycle engines	270	32
Car SI engines	250	35
Indirect injection car diesel engines	240	35
Turbocharged DI car diesel engines	200	42
Turbocharged truck diesel engines	190	45
Crosshead engines (two-stroke diesel)	156	54



Specific power, power to weight

Engine type	Specific power output [kW/l] up to	Power-to-weight ratio [kg/kW] up to	At engine speed [rpm]
Racing engine (Formula 1)	200	0.4	($n \approx 18\,000$ rpm)
Car SI engine	70	2.0	($n \approx 6\,500$ rpm)
Turbocharged car SI engine	100	3.0	($n \approx 6\,000$ rpm)
Car diesel engine (naturally aspirated)	45	5.0	($n \approx 4\,500$ rpm)
Turbocharged car diesel engine	64	4.0	($n \approx 4\,500$ rpm)
Commercial vehicle diesel engine	30	5.5	($n \approx 3\,000$ rpm)
High-speed diesel engine	15.0	11.0	($n \approx 4\,500$ rpm)
Medium-speed diesel engine	7.5	19.0	($n \approx 500$ rpm)
Slow large diesel engine (two-stroke)	3.0	55.0	($n \approx 100$ rpm)

Volumetric efficiency

Volumetric efficiency λ_l (or η_{vol}) is a measure for the charge cycle and it tells how much fresh charge has been trapped in the cylinder during charge cycle

$$\lambda_l = \frac{m_{Zges}}{\rho_{th} V_H}$$

m_{Zges} is the mass of charge air delivered to the cylinder and

V_H is the total displacement volume of the engine. ρ_{th} is the density of outside air. Volumetric efficiency is a very important value for naturally aspirated SI engines. The better the volumetric efficiency, the greater the maximum torque.

Efficiency and fuel consumption

Total efficiency of an engine is the ratio of the brake power and the energy content of fuel flow

P_e is brake power

\dot{m}_K is fuel mass flow

H_u is fuel net heating value

Specific fuel consumption is the ratio of fuel mass flow and brake power

Hence we obtain a relation between specific fuel consumption and total efficiency

$$\eta_e = \frac{P_e}{\dot{m}_K H_u}$$

$$b_e = \frac{\dot{m}_K}{P_e}$$

$$\eta_e = \frac{1}{b_e H_u}$$

Mechanical efficiency

Mechanical efficiency is the ratio of the brake power flow to power of gases working against piston

Mechanical efficiency is also the ratio of brake mean effective and indicated mean effective pressure

$$\eta_{mech} = \frac{P_e}{P_i} = \frac{BMEP}{IMEP}$$

