



Aalto University
School of Engineering

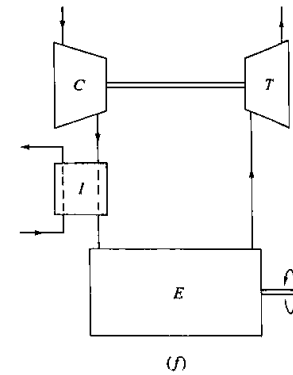
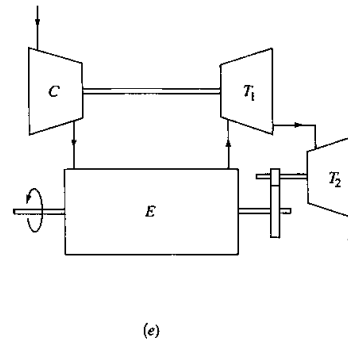
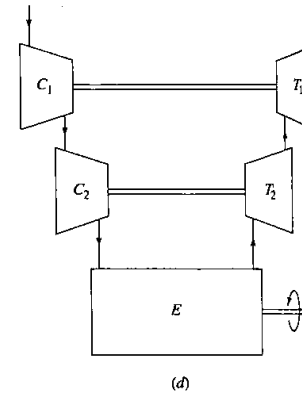
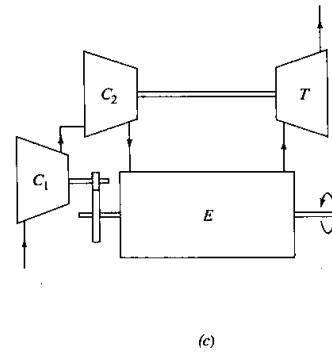
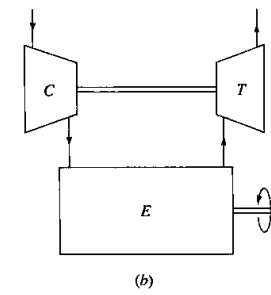
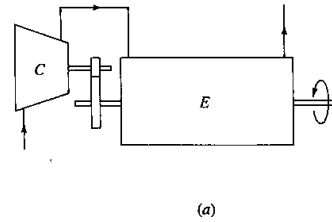
EEN-E2002, Internal Combustion Air Management 2, supercharging

Basshuysen Chapter 11 Supercharging of Internal Combustion Engines
Heywood Chapter 6 Gas exchange process

January 2019, Martti Larmi

The six basic ways of supercharging and turbocharging, Heywood Fig. 6-37

- C Compressor
- T Turbine
- E Engine
- I Charge air cooler / Inter cooler



Adiabatic flow in during compression

$$h_0 = h + \frac{C^2}{2}$$

Subscript 0 refers to total conditions, i.e. dynamic effects included

$$T_0 = T + \frac{C^2}{2c_p}$$

$$p_0 = p \left(\frac{T_0}{T} \right)^{\gamma/(\gamma-1)}$$

$$-\dot{W} = \dot{m}(h_{0,\text{out}} - h_{0,\text{in}})$$

Isentropic efficiency

Subscript 1 refers to conditions before compressor and subscript 2 after the compressor, s refers to isentropic

$$\eta_c = \frac{\text{reversible power requirement}}{\text{actual power requirement}}$$

$$\eta_{CTT} = \frac{(p_{02}/p_{01})^{(\gamma-1)/\gamma} - 1}{(T_{02}/T_{01}) - 1}$$

C compressor

$$\eta_{CTS} = \frac{T_{2s} - T_{01}}{T_{02} - T_{01}} = \frac{(p_2/p_{01})^{(\gamma-1)/\gamma} - 1}{(T_{02}/T_{01}) - 1}$$

TT total to total

TS total to static

$$-\dot{W}_C = \dot{m}_i c_{p,i} (T_{02} - T_{01}) = \frac{\dot{m}_i c_{p,i} T_{01}}{\eta_{CTT}} \left[\left(\frac{p_{02}}{p_{01}} \right)^{(\gamma-1)/\gamma} - 1 \right]$$

Isentropic and mechanical efficiency

$$\eta_t = \frac{\text{actual power output}}{\text{reversible power output}}$$

Subscript 3 refers to conditions before turbine and subscript 2 after the turbine

$$\eta_{TTT} = \frac{1 - (T_{04}/T_{03})}{(p_{04}/p_{03})^{(\gamma-1)/\gamma} - 1}$$

$$\eta_{TTS} = \frac{T_{03} - T_{04}}{T_{03} - T_{04s}} = \frac{1 - (T_{04}/T_{03})}{(p_4/p_{03})^{(\gamma-1)/\gamma} - 1}$$

T turbine

TT total to total

TS total to static

$$-\dot{W}_T = \dot{m}_e c_{p,e} (T_{03} - T_{04}) = \eta_{TTT} \dot{m}_e c_{p,e} T_{03} \left[1 - \left(\frac{p_{04}}{p_{03}} \right)^{(\gamma-1)/\gamma} \right]$$

$$-\dot{W}_C = \eta_m \dot{W}_T$$

Superchargers

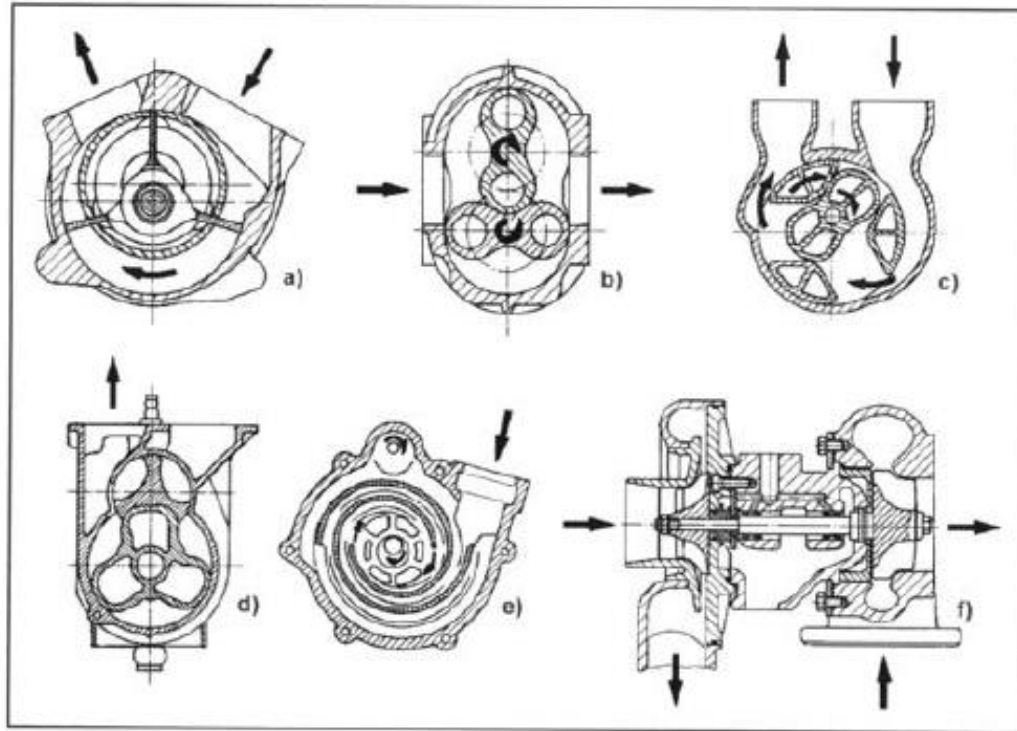
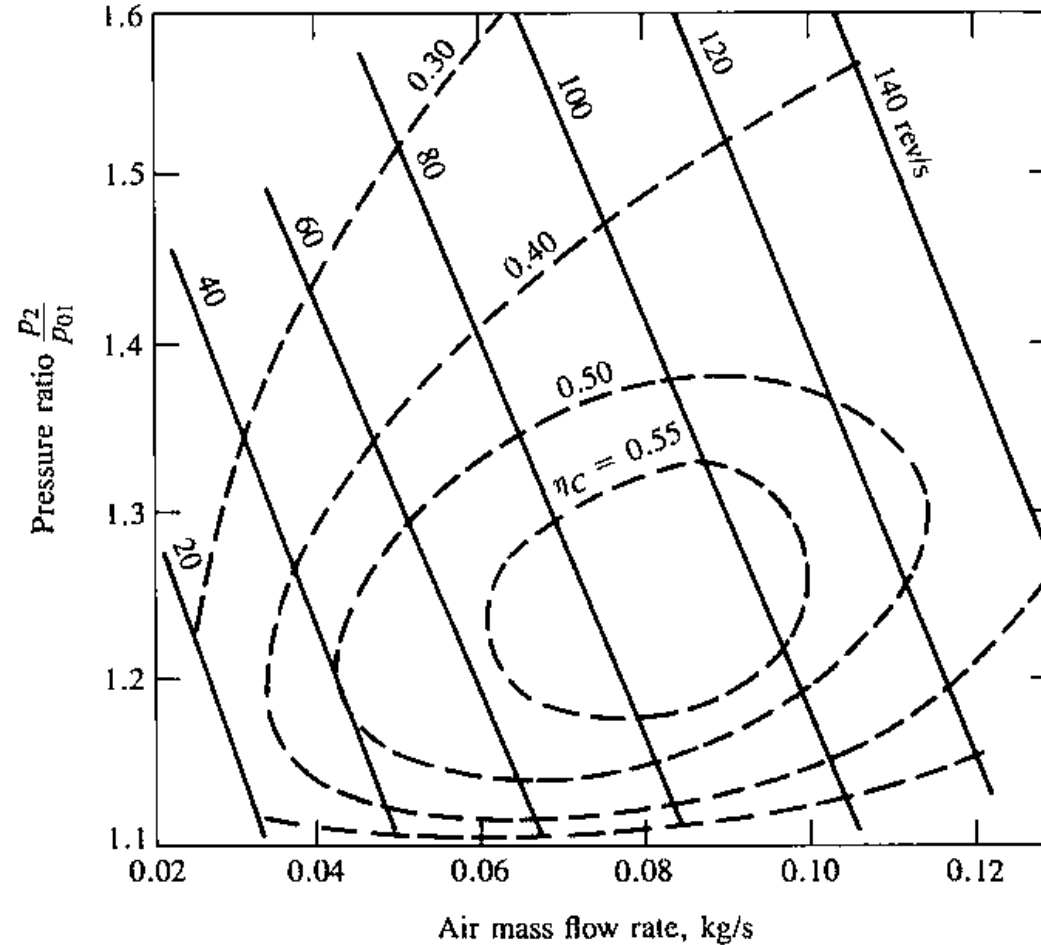
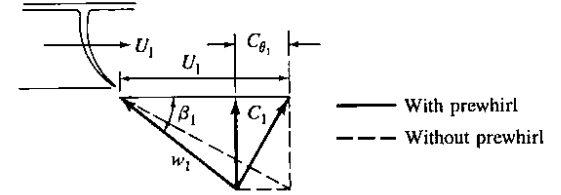
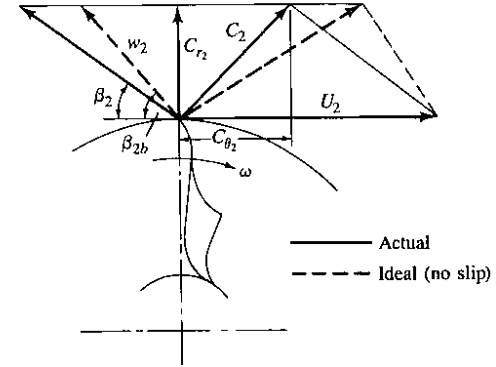
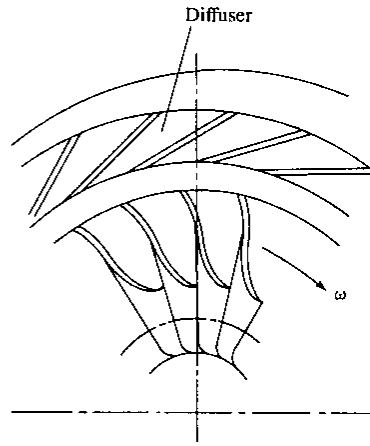
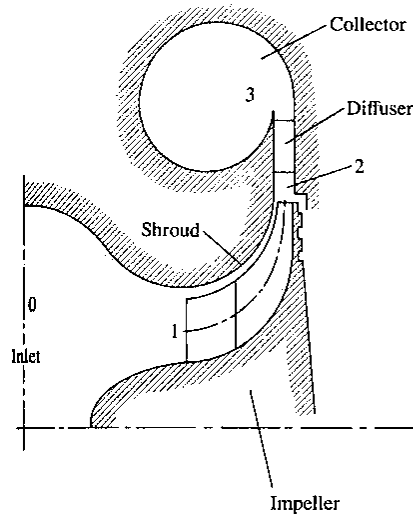


Fig. 10-60 Overview of the different designs of blowers and superchargers: (a) Vane-type superchargers, (b) Roots superchargers, (c) Rotary piston superchargers, (d) Screw compressors, (e) Spiral superchargers (G-superchargers), (f) Turbochargers.

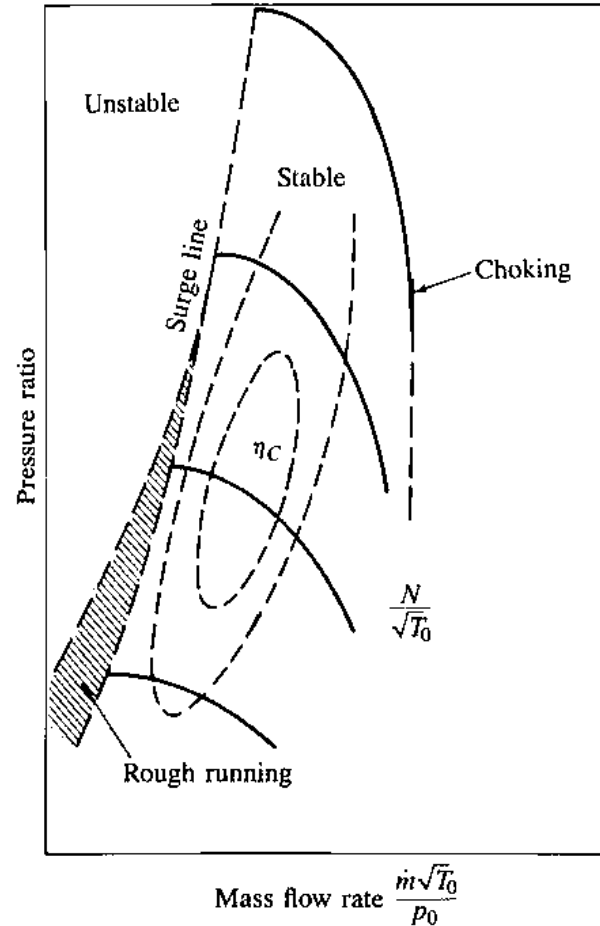
Mechanical compressor map



Turbocharger compressor



Turbocharger compressor map 1



Turbocharger compressor map 2

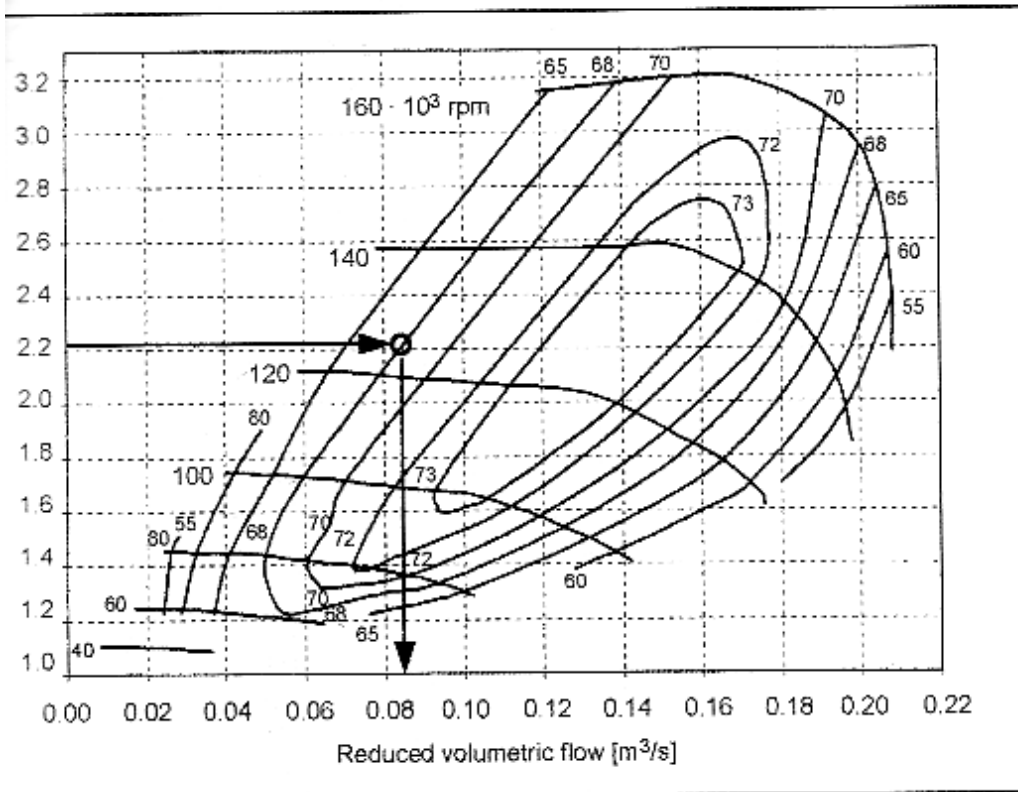
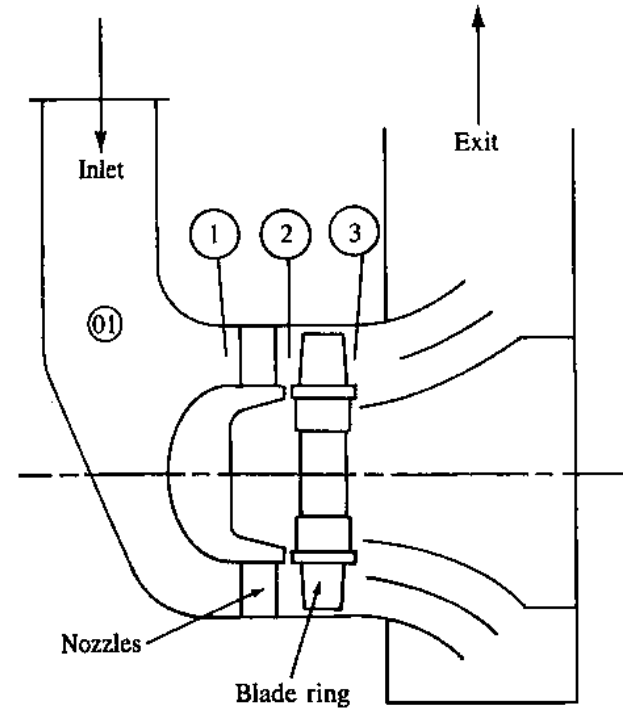
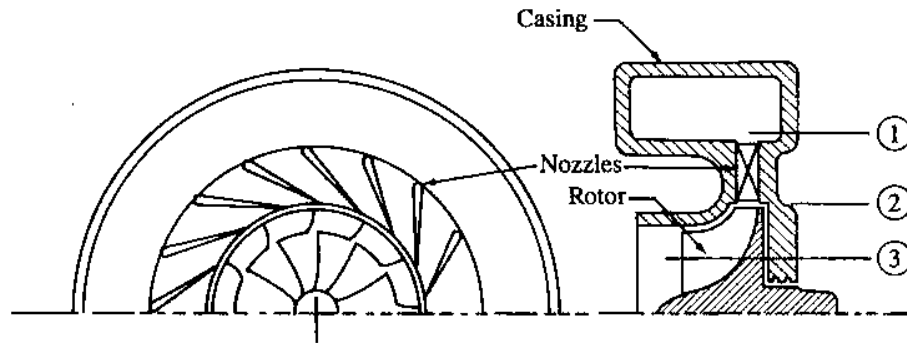
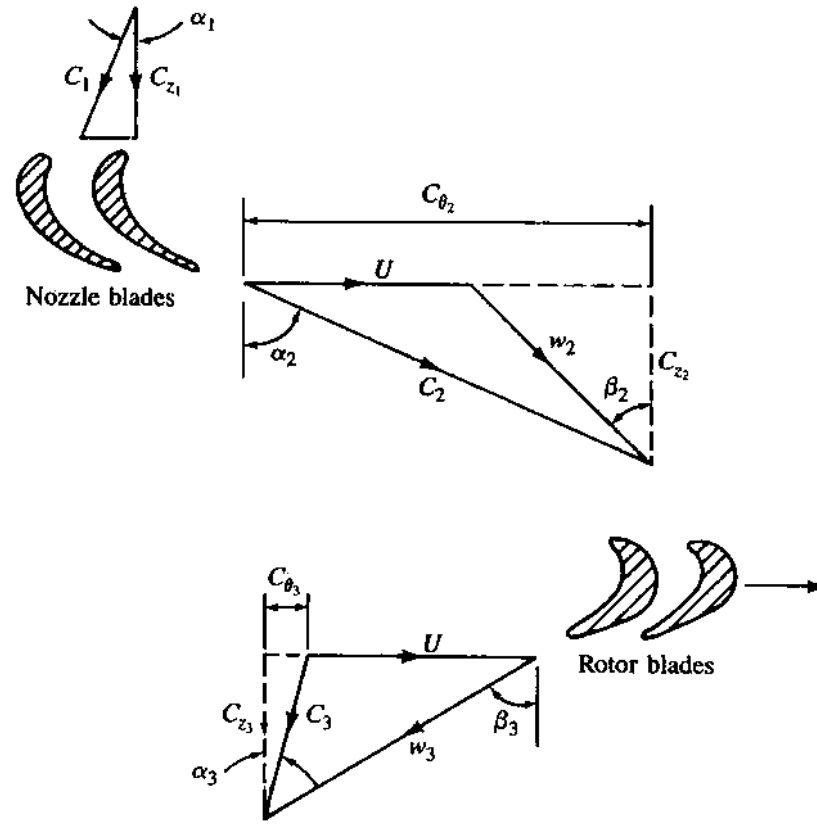


Fig. 11-9 Compressor map.

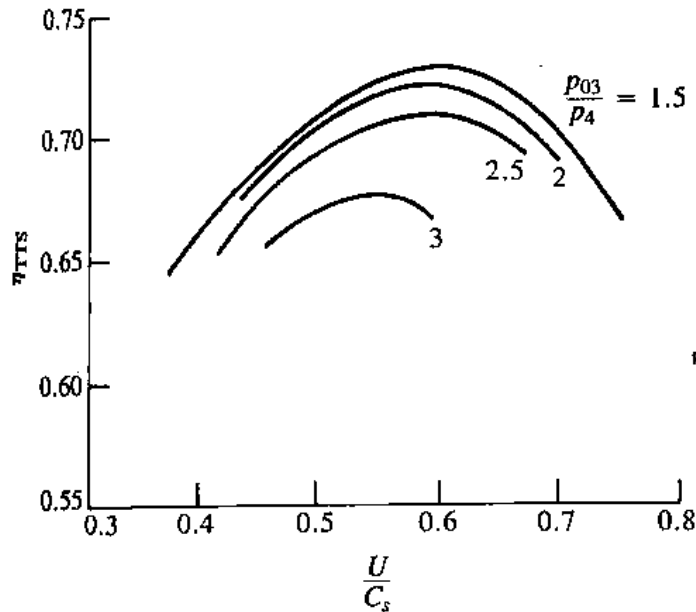
Turbocharger radial turbine and axial turbine



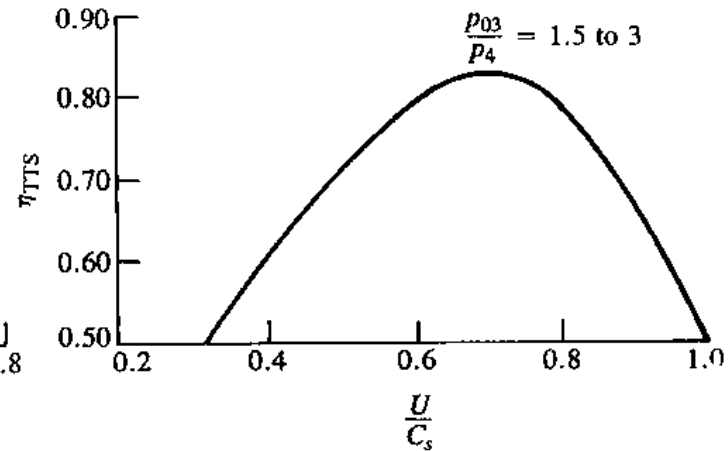
Velocity triangles of an axial turbine



Turbine isentropic efficiency and blade speed ratio U/C_s

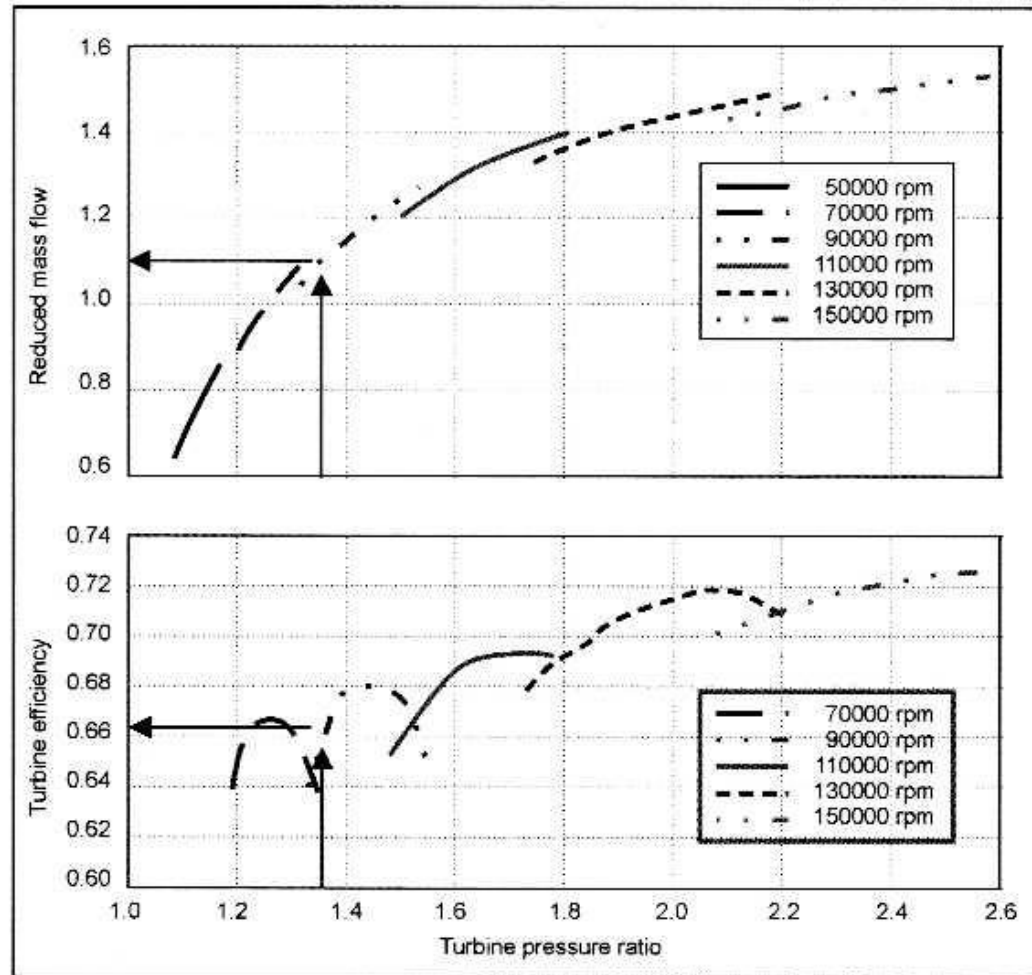


(a) Axial flow



(b) Radial flow

Turbine map



Gas exchange and turbine loading

Turbine loading:

- a) Pulse turbocharging: narrow piping, unsteady flow, high pulses, fast reaction
- b) Constant pressure turbocharging: large volume in exhaust system, steady flow in turbine, slow reactions, high turbine efficiency

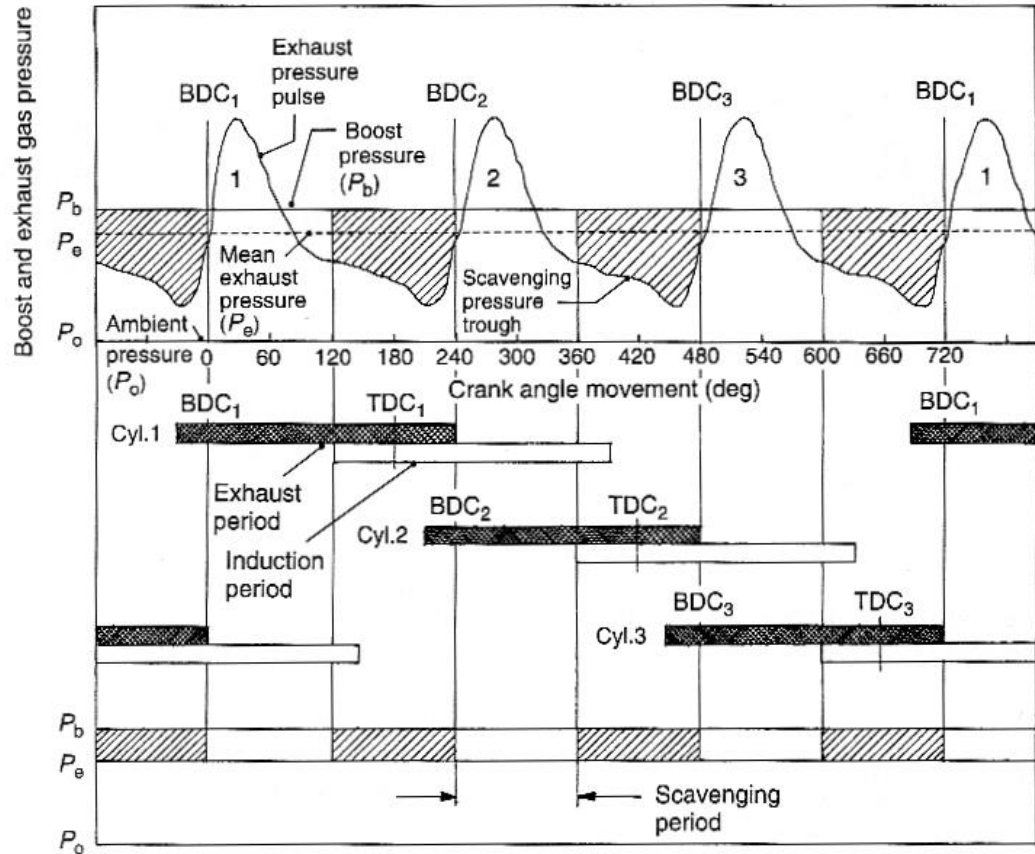
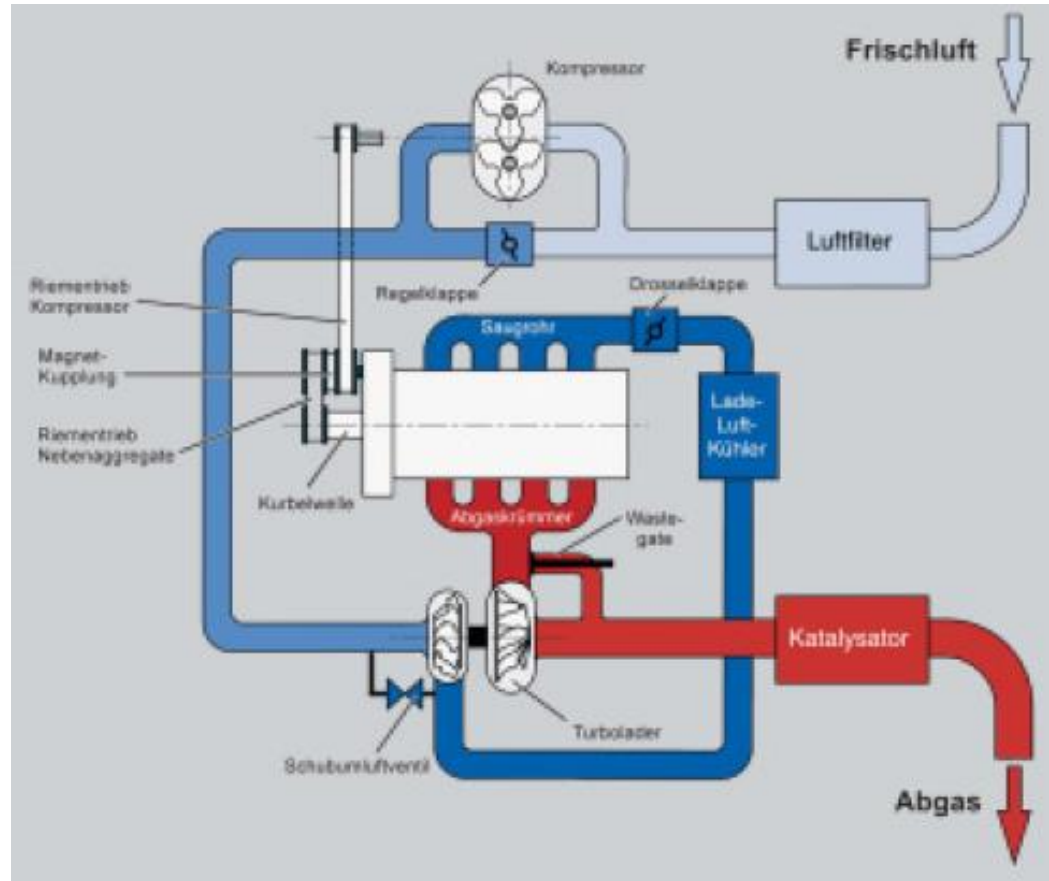


Fig. 6.47 Exhaust gas pressure variation in activated six-cylinder turbocharged engine manifold

VW TSI Charging system



Flow through the engine and flow through the compressor

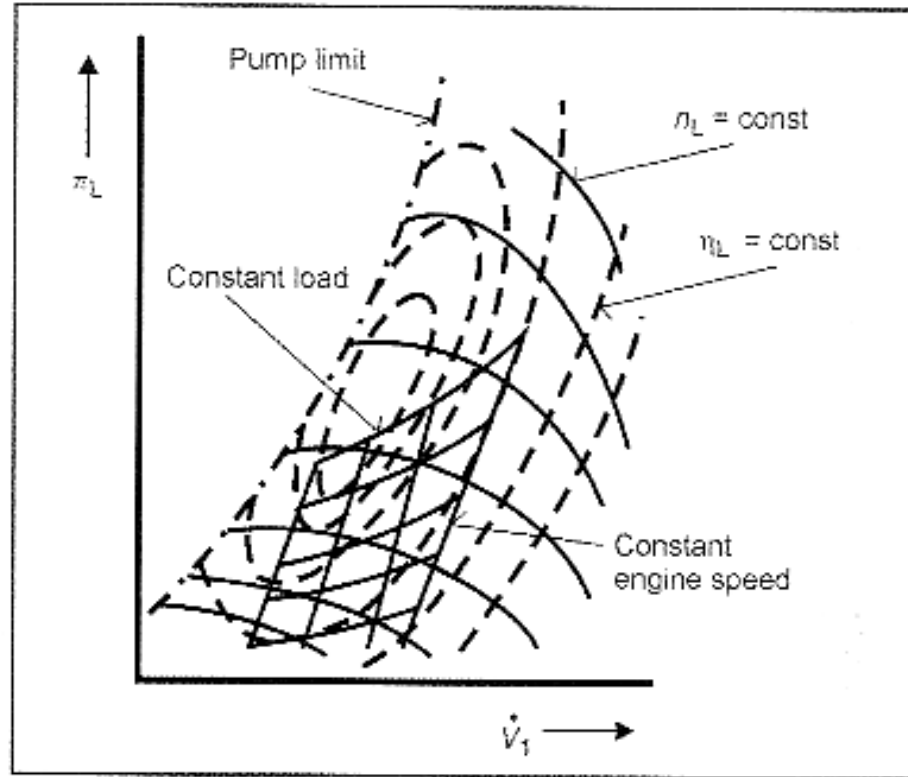


Fig. 11-26 Superimposition of maps.