

Aalto University School of Electrical Engineering

Communication acoustics Ch 2: Physics of sound – Acoustics

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Physics of sound

- Basic quantities
- Vibration → generation of sound
- Resonance, resonators
- Sound radiation
- Sound propagation
- Reflection, absorption,
- Diffraction, refraction
- Modeling of acoustics

Sound pressure

- Atmospheric pressure $p_a \approx 101$ kPa, can be assumed constant with time
- Pressure disturbances travel as sound waves

Total pressure
$$p_{tot(t)} = p_a + p(t)$$

- Instantaneous sound pressure p(t) [Pa] can be captured with a microphone
- Audible sounds have frequencies between about 20Hz and 20kHz
- Effective value (rms value) of sound pressure

$$p_{\rm rms} = \sqrt{\frac{1}{t_2 - t_1} \int_{t_1}^{t_2} p(t)^2 dt}$$

Sound pressure level, Decibel

- Effective sound pressure p [Pa] (rms value)
- Sound pressure level $L_p = 20 \log_{10}(p/p_0)$
- Reference pressure $p_0 = 20 \cdot 10^{-6}$ Pa

ratio	decibels	ratio	decibels
1/1	0		
$\sqrt{2} pprox 1.41$	pprox 3.01 $pprox$ 3	$\sqrt{1/2}pprox 0.71$	pprox -3.01 pprox -3
2/1	pprox 6.02 $pprox$ 6	1/2	pprox -6.02 pprox -6
$\sqrt{10} pprox 3.16$	10	$\sqrt{1/10}pprox 0.316$	-10
10/1	20	1/10	-20
100/1	40	1/100	-40
1000/1	60	1/1000	-60

Vibration – generation of sound

- Most of physical sounds in natural environment are caused by vibrating objects
- Frequency range 20 Hz 20 kHz (audible frequencies)
- Impact sounds, water, animal sounds, human voice, musical instruments
- Exception: electric sparks, thunder



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Vibrating systems



Simple vibration: mass-spring system

Vibrating systems



•
$$\mathbf{y}(t) = \mathbf{A}\mathbf{e}^{-\alpha t}\cos(\omega_{\mathrm{d}}t + \phi) = \mathbf{A}(t)\cos(\psi(t))$$

Resonance



Mass-spring resonator

Single mass, single resonance, single mode

Resonance



Helmholtz-resonator

Two-mass vibrating systems



- Transversal and longitudinal vibration of a two-mass system
- Each case forms a mode (a resonance)
- Two modes in transversal vibration
- Two modes in longitudinal vibration

Three-mass vibrating systems



Transversal and longitudinal vibration of a three-mass system

Vibration modes of a string



String is continuous, infinite number of masses

- Infinite number of modes, whose resonance frequencies are integer multiples of fundamental frequency
- Harmonic spectrum
- (demo with guitar string)

Modes and spectral content in string vibration



Modes in bars



- Stiffness of bar makes frequencies to travel with different speeds
- Modes are not related to each other with integer relations
- Inharmonic spectrum
- (demo)

Resonance modes in tube



- Harmonic spectrum
- Spectrum: a) all harmonics b) only odd harmonics
- (demo with tube)

1D wave propagation

- Wave equation $\ddot{y} = c^2 y''$
- D'Alember 1D solution $y(t, x) = g_1(ct x) + g_2(ct + x)$

 $\bullet \ \lambda = \mathbf{c}/\mathbf{f}$



Wave propagation animations

Click for animation

Animation courtesy of Dr. Dan Russell, Grad. Prog. Acoustics, Penn State

Speed of wave propagation c

Wave phenomena: plane wave in tube



Reflection and transmission. $R = \frac{Z_2 - Z_1}{Z_2 + Z_1}$



Modes in 2D membranes



 \blacksquare Complex distribution of modes \rightarrow inharmonic spectrum

Radiation from sound source

- Sound is caused by mechanical vibrations in audible range of frequencies
- Sound source is coupled to air
- Some of energy of source vibration emanates as sound
- Radiation has often directional pattern, sound is radiated to different directions with different strengths

Spherical wave propagation

- Sound speed in air $c_{air}(T) = 331.3 + 0.6T$
- Longitudinal wave, moving rarefactions and compressions
- Spherical wave:
 - Energy is constant in each spherical wave
 - Area of the wave $\propto r^2 \rightarrow$ energy decays with $1/r^2$ law
 - Amplitude decays with 1/r law
 - $p_1r_1 = p_2r_2, p_2 = p_1(r_1/r_2)$



Directional patterns

- Omnidirectional radiates with amplitude coefficient c₀ to all directions
- Dipole $c_1 \cos \theta$
- Quadrupole c₂ cos 2θ
- When source is large compared to wave length, the radiation pattern is affected a lot
- The directionality is often irregular, as sources are typically irregular in shape

Link to dipole radiation video

Link to video on acoustic directivities

Reflection and refraction

- After radiation, sound wave continues straight ahead, if medium is still and has constant density
- In cases of obstacles, or changes in medium, sound changes its direction





Diffraction



 non-planar surfaces and especially edges (other than 90 degrees corners) act as secondary sources

Sound propagation paths in a room



Impulse response of a room



- Theoretic response to an ideal impulse
- Instantaneous amplitude of p(t) is plotted

Impulse response of a room



Real measured response of a listening room to a laser-induced spark source

3D propagation of sound visualized in 2D plane

Link to ripple tank visualization

Diffuse field, room modes

- Diffuse field: sound with equal frequency content arrives evenly from all directions with random phase relations
- Late reverberation produces diffuse field in many rooms
- Rooms have also resonances
- Standing waves can be evoked with sinusoidal stimulus
- Room modes: spatial distribution of pressure [or velocity] maxima

Link to room mode visualization

Modes in rectangular room

- A normal room has its own resonances
- Mode frequencies for a room with only right-angled corners



Reverberation time

- The time that it takes sound to decay 60dB after offset
- Can be measured from impulse response
- In reverberant rooms similar values found for different positions
- In rooms with absorbents, value may change a lot depending on position



Estimation of reverberation time

- Sabine's formula for simple estimation from room geometry: $T_{60} = 0.161 V/S$
- Volume of room V [m³]
- Absorption area of room S
- $S = \sum a_i A_i$, a_i is absorption coefficient (e.g., table below)
- A_i is surface area [m²]
- Assumes diffuse field: not accurate often (non-cubical rooms, non-equal distribution of absorption)

Frequency	125	250	500	1000	2000	4000
Glass window	0.35	0.25	0.18	0.12	0.07	0.04
Painted concrete	0.10	0.05	0.06	0.07	0.09	0.08
Wooden floor	0.15	0.11	0.10	0.07	0.06	0.07

Level of sound field in room as function of distance from source



Sound pressure caused by multiple sources

- Sound pressure is always measured in single position with a microphone (or listened to with ear)
- Often microphone captures sound (almost) equally from all directions
- How can we compute the sound pressure caused by multiple sources?

$$p_{\rm rms} = \sqrt{\frac{1}{t_2 - t_1} \int (p_1(t) + p_2(t))^2 dt}$$

$$p_{\rm rms} = \sqrt{rac{1}{t_2 - t_1}} \int (p_1^2(t) + 2p_1(t)p_2(t) + p_2^2(t)) {
m d}t$$

 $2p_1(t)p_2(t)$ has mean value of zero if $p_1(t)$ and $p_2(t)$ are uncorrelated

$$p_{\rm rms} = \sqrt{\frac{1}{t_2 - t_1} \int (p_1^2(t) + p_2^2(t)) dt}$$

Sound pressure caused by multiple sources

 If coherent sound arrives from multiple directions to microphone (reflections, stereophonic sound)

- Two sources: $p_{tot}(t) = p(t) + p(t) = 2p(t)$
- 6dB increase in prms
- If incoherent sound arrives from multiple directions to microphone (multiple concurrent sources)
 - N sources: $p_{\text{tot}_{\text{rms}}} = \sqrt{\sum p_{\text{n}_{\text{rms}}}^2}$ Two sources: $L_{\text{tot}} = 10 \log_{10} \left(10^{L_1/10} + 10^{L_2/10} \right)$ 3dB increase in p_{rms}

Example 1, radiation from loudspeaker cone

A loudspeaker radiates sound with good efficiency, if the dimensions of the radiating surface are of the same order with sound being radiated. Too small radiating surface causes only air movement in vicinity of loudspeaker, and no sound is radiated far field. What can you say about radiation from a loudspeaker cone? What diameter should be needed for radiation at 30Hz, 1kHz or 2kHz.

 $\lambda = \mathbf{c}/\mathbf{f}$

Example 1

Lets assume c = 331.5 m/s $\lambda = c/f$

30Hz: $\lambda = \frac{331.5 \text{m/s}}{30 \text{ I/s}} = 11.05 \text{m} \approx 11 \text{ m}.$

Example 1

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30Hz: $\lambda = \frac{331.5 \text{m/s}}{30 \text{ l/s}} = 11.05 \text{m} \approx 11 \text{ m}.$

1kHz: $\lambda = \frac{331.5 \text{m/s}}{1000 \text{ 1/s}} = 0.332 \text{ m} \approx 33 \text{ cm}.$

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1kHz:
$$\lambda = rac{331.5 {
m m/s}}{1000 {
m 1/s}} =$$
 0.332 m $pprox$ 33 cm.

10kHz:
$$\lambda = \frac{331.5 \text{m/s}}{10000 \text{ 1/s}} = 0.0332 \text{ m} \approx 3.3 \text{ cm}.$$

In practise, even largest cones are too small for 30Hz.

Example 2, loudspeaker cone coupled to a vented box

The efficiency of a cone in free air can be made better by attaching it to a box, and even better radiation is obtained when the resonance frequency of the box adjusted to match with desired frequency. A loudspeaker with bass reflex principle is in fact a Helmholtz resonator.

A typical small loudspeaker has volume *V* of 5500 cm³, and a reflex tube with length *I* and opening area of a=8cm². What would be the correct tube length for 45Hz resonance frequency? Helmholtz resonance frequency can be computed

$$f = rac{c}{2\pi} \sqrt{rac{a}{VI}} \Leftrightarrow I = a rac{c^2}{V4\pi^2 f^2}$$

Example 2, loudspeaker cone coupled to a vented box

Helmholtz resonance frequency can be computed

$$f = rac{c}{2\pi} \sqrt{rac{a}{VI}} \Leftrightarrow I = a rac{c^2}{V4\pi^2 f^2}$$

Numerical values:

$$I = 8 * \frac{(340 * 100 \text{ cm/s})^2}{5500 * 4\pi^2 45^2} = 21 \text{ cm}$$



The dimensions of an empty right-angled room are $10 \text{ m} \times 6 \text{ m} \times 3 \text{ m}$. The absorption coefficients at 500Hz are in floor 0.06, ceiling 0.17 and walls 0.20. There are no windows neither doors in the room.

1. Compute the T_{60} of the room

2. What absorption coefficient should the walls have to obtain 0.7s for T_{60} ? Sabine's formula to estimate (more or less roughly) reverberation time:

$$T_{60} = 0.161 \frac{V}{S}$$

where V = volume, S = absorption area = $\sum_i a_i A_i$, A_i = area of a surface i, and a_i = absorption coefficient of surface i

Volume of the room: $V = 10 * 6 * 3 = 180 \text{ m}^3$

1. Reverberation time. Total absorption area: $S = S_f + S_c + S_w$

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 $T_{60} = 0.161 \frac{V}{S} = 0.161 * \frac{180}{33} = 0.878 \text{ s}$

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2. Absorption coefficient for walls. Absorption area: $S_w = K \frac{V}{T_{60}} - S_f - S_c$

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2. Absorption coefficient for walls. Absorption area: $S_w = K \frac{V}{T_{60}} - S_f - S_c$ Substitute numeric values $S_w = 0.161 \frac{180}{0.7} - 3.6 - 10.2 = 27.6 \text{ m}^2$

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2. Absorption coefficient for walls.

Absorption area: $S_w = K \frac{V}{T_{60}} - S_f - S_c$ Substitute numeric values $S_w = 0.161 \frac{180}{0.7} - 3.6 - 10.2 = 27.6 \text{ m}^2$ The absorption coefficient should then be: $a = \frac{S_w}{A_w} = \frac{27.6}{2*30+2*18} = 0.29$

Average talking produces 60dB SPL in 1m distance. What would be the SPL when

- 1. two persons are talking?
- 2. 10 persons are talking?
- 3. two persons with distances 1m and 3m are talking?

Assume that all of them are in 1m distance from the position of measurement.

Sound arriving from each source is incoherent, which means that the effective pressure p will be summed as quadratic.

$$p=\sqrt{(p_1^2+p_2^2)}$$

Effective sound pressure caused by one talker is p_1 . Lets compute SPLs:

1. Two talkers

The pressure for two talkers is $p_2 = \sqrt{p_1^2 + p_1^2} = p_1 \sqrt{2}$. \Rightarrow

 $L = 20 \lg(p_2) = 20 \lg(p_1 \sqrt{2}) = L_1 + 3.01 dB = 63 dB.$

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- **2.** Ten talkers: $\Rightarrow L = 20 \lg(\sqrt{10} * p_1) = L_1 + 10 \text{ dB} = 70 \text{ dB}$

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- **2.** Ten talkers: $\Rightarrow L = 20 \log(\sqrt{10} * p_1) = L_1 + 10 \text{ dB} = 70 \text{ dB}$
- 3. Two talkers with distances of 1 and 3 meters: $p_t = \sqrt{p_1^2 + (p_1(r_1/r_2))^2} = \sqrt{p_1^2 + (p_11/3)^2} = \sqrt{p_1^2 + 1/9(p_1)^2} = p_1\sqrt{10/9}$

$$L = 20 \lg(\sqrt{10/9} * p_1) = L_1 + 0.5 dB = 60.5 \text{ dB}$$
(2)

References

These slides follow corresponding chapter in: Pulkki, V. and Karjalainen, M. Communication Acoustics: An Introduction to Speech, Audio and Psychoacoustics. John Wiley & Sons, 2015, where also a more complete list of references can be found.



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