

Sound Radiation from Vibrating Objects

ELEC-E5610 Acoustics and the Physics of Sound, Lecture 10

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General

For finite vibrating objects, the sound field created by the vibration is affected by



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 - amplitude
 - phase
 - frequency



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- surrounding environment



General

For finite vibrating objects, the sound field created by the vibration is affected by

- the shape and dimensions of the surface
- vibrational distribution on the surface
 - amplitude
 - phase
 - frequency
- surrounding environment
- the interaction of all sides of the vibrating surface





Circular Piston

Pressure Field

Intensity Field

Scalability

Directivity Approximations

Radiation Impedance Consider a circular piston with radius R as a sound source



- Pressure Field
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- Radiation Impedance

Consider a circular piston with radius R as a sound source

- vibrates as a single unit with velocity \tilde{u}_{n}
 - constant amplitude, phase, and frequency



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 - constant amplitude, phase, and frequency
- placed on an infinite rigid plane
 - no interaction from the other side of the plane



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Approximation of a baffled loudspeaker! Let's study the radiation pattern more thoroughly...



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Radiation Impedance The far-field sound pressure created by the piston source is

$$\tilde{p}(r,\theta) = \frac{i\rho\omega R^2 \tilde{u}_n}{2r} \left[\frac{2J_1(kR\sin\theta)}{kR\sin\theta} \right] e^{-ikr}$$
(FF:6.53,R&F:7.30)

where J_1 is the 1st-order Bessel function.

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where J_1 is the 1st-order Bessel function. Compare to the far-field sound pressure of the monopole ($q_0 = 4\pi R^2 \tilde{u}_n$):

$$\tilde{p}(r) = \frac{i\omega\rho R^2 \tilde{u}_n}{r} e^{-ikr}$$
(FF:6.20,R&F:7.4)



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They are the same, except for scaling by $\frac{1}{2}$



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(FF:6.20,R&F:7.4)

They are the same, except for scaling by $\frac{1}{2}$ and a multiplication term, called the **directivity function**.



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Fig. 6.17 Radial sound intensity distribution in the far field of a circular piston in a rigid baffle at (a) ka = π/4; (b) ka = π; (c) ka = 4π. Reproduced with permission from Kinsler, L. E., Frey, A. R., Coppens, A. B. and Sanders, J. V. (1982) *Fundamentals of Acoustics*, 3rd edn. John Wiley & Sons, New York.



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	In other words, pressure field $=$ pressure field of a monopole	
Circular Piston	\times directivity function $f(\theta) = \left[\frac{2J_1(kR\sin\theta)}{2}\right]$	
Pressure Field	$\wedge \operatorname{directivity} \operatorname{function} f(\theta) = \begin{bmatrix} kR\sin\theta \end{bmatrix}$.	$\begin{bmatrix} kR\sin\theta \end{bmatrix}$
Intensity Field	$f(\theta)$ gives	0° 10°
Scalability	the pressure	
Directivity Approximations	magnitude	(c) 30"
Radiation Impedance	as a function of θ	(b)

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1 90'



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-10

+10 0 log I

	In other words, pressure field = pressure field of a monopole	
Circular Piston	\times directivity function $f(\theta) = \left[\frac{2J_1(kR\sin\theta)}{2\pi i \pi^2}\right]$	
Pressure Field	$KR\sin\theta$	
Intensity Field	$f(\theta)$ gives	
Scalability	the pressure	
Directivity Approximations	magnitude	
Radiation Impedance	as a function of θ	
	$f(\theta) \text{ also} \\ \text{frequency-} \\ \text{dependent,} \\ \text{since } k = \\ 2\pi/\lambda = \omega/c \end{cases}$ $F_{g. 6.17 Radial sound intensity distribution in the far field of a circular piston in a rigid baffle at (a) ka = \pi/c (b) ka = \pi/$	



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2 Intensity Field Generated by a Circular Piston



Figure: Active intensity for (a) low and (b) high frequencies (FF:p.129)



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2 Scalability

Circular Piston

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Intensity Field

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Directivity Approximations

Radiation Impedance As in the piston case, the directivity patterns of sound sources in general are not dependent on absolute dimensions, but rather on kR, called the **Helmholtz** number



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sound field behaves similarly when the wavelength changes proportionally to the object dimensions

 \blacksquare \Rightarrow enables the use of acoustic scale models!



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- sound field behaves similarly when the wavelength changes proportionally to the object dimensions
- \Rightarrow enables the use of acoustic scale models!

However, scale models should be used carefully if significant losses are present, for example

- with very high frequencies
- small ducts, porous materials, etc.



2 Scalability II

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Figure: Scale model of a concert hall



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2 Directivity Approximations

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Radiation Impedance When increasing $kR \sin \theta$ from zero, we find out that the $f(\theta) = \left[\frac{2J_1(kR \sin \theta)}{kR \sin \theta}\right]$ has its first zero when $kR \sin \theta \approx 3.83$.



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 \blacksquare \Rightarrow low frequencies radiate to all directions!



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In fact, for $kR \ll 1$ (i.e. if the dimensions of the piston are very small compared to wavelength)

 $\Rightarrow f(\theta) = \left[\frac{2J_1(kR\sin\theta)}{kR\sin\theta}\right] \approx 1$, so that the piston source approximates an elementary monopole.



Circular Piston

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Radiation Impedance The mechanical radiation impedance of a circular piston (relation between piston velocity and the resulting force exerted by the fluid to the piston) is

$$z_{\rm mrad} \approx \rho c \pi R^2 \left[\frac{(kR)^2}{2} + i \frac{8kR}{3\pi} \right]$$
 (FF:(6.55))

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- on the surface of the piston: piston velocity = particle velocity of air
- Eq. (FF:(6.55)) is a low kR approximation. Let's see what the exact ^Zmrad ρcπR² looks like...





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Fig. 6.19 Radiation impedance ratios of baffled and unbaffled pistons. Radiation impedances as follows: (1) $X_{rad}/\pi a^2 \rho_0 c$ for an unbaffled piston; (2) $X_{rad}/\pi a^2 \rho_0 c$ for a baffled piston; (3) $R_{rad}/\pi a^2 \rho_0 c$ for an unbaffled piston; (4) $R_{rad}/\pi a^2 \rho_0 c$ for a baffled piston; (3) Reginder the form Rschevkin, S. N. (1963) Lectures on the Theory of Sound. Pergamon Press, Oxford.

Figure: Real R and imaginary X components of rad. impedance



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What can you say about the low-frequency-response of the baffled vs. unbaffled piston?



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Figure: Real R and imaginary X components of rad. impedance

What can you say about the low-frequency-response of the baffled vs. unbaffled piston? What about high frequencies?



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Radiation Impedance Let's take a look at the reactive part of Eq. (FF:(6.55)):

$$\mathsf{m}[z_{\mathrm{mrad}}] pprox
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remember $\mathbf{k} = \omega/\mathbf{c} \Rightarrow$

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How would you interpret the reactive radiation impedance for the piston?



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$$\operatorname{Im}[z_{\mathrm{mrad}}] \approx i\omega \frac{8\rho R^3}{3}.$$

How would you interpret the reactive radiation impedance for the piston? It represents an attached mass of $\frac{8\rho R^3}{3}$ kg.



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Radiation Impedance Since FF:Fig.6.19 showed that the bracketed resistive part of $z_{mrad} \approx 1$ for high frequencies, it can be concluded that

$$\operatorname{Re}[z_{\mathrm{mrad}}] \approx \pi R^4 \rho c$$



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How would you interpret this?



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How would you interpret this? Remember the characteristic impedance of a plane wave $z_c = \rho c$. This means that for high frequencies, the piston behaves as a plane wave source.



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How would you interpret this? Remember the characteristic impedance of a plane wave $z_c = \rho c$. This means that for high frequencies, the piston behaves as a plane wave source. See also https://www.acs.psu.edu/drussell/Demos/BaffledPiston/BaffledPiston.html.







Radiator Groups

Sound Pressure Directivity Patterns Wavefield Synthesis Consider a case where a group of equal-phase monopoles are placed on the *z*-axis, *d* meters apart.

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Radiator Groups

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P is the observation point at a distance r from the group midpoint





Radiator Groups

Sound Pressure Directivity Patterns Wavefield Synthesis Consider a case where a group of equal-phase monopoles are placed on the *z*-axis, *d* meters apart.

- P is the observation point at a distance r from the group midpoint
- θ is the angle between P, group midpoint, and z-axis





Radiator Groups

Sound Pressure

Directivity Patterns Wavefield Synthesis

The sound pressure at P caused by N monopoles can be given as

$$\tilde{\rho}(\theta, r) \approx \left(\frac{i\omega\rho q_0}{4\pi r}\right) e^{-ikr} \left[\frac{\sin\left(\frac{N\pi d}{\lambda}\cos\theta\right)}{\sin\left(\frac{\pi d}{\lambda}\cos\theta\right)}\right] \quad (\mathsf{R}\&\mathsf{F}:(7.17))$$



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sound pressure of a monopole

directivity function



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which consists of

sound pressure of a monopole

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(note that book version of R&F:7.17 uses a number of 2N monopoles)



3 Directivity Patterns

Radiator Groups

Sound Pressure

Directivity Patterns

Wavefield Synthesis

Figure: directivity patterns created by 7 point sources (illustrated with red circles)



Fig. 2. Simulated far-field polar response of a 7-pointsource uniform array.



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3 Directivity Patterns

- **Radiator Groups**
- Sound Pressure
- Directivity Patterns
- Wavefield Synthesis

Figure: directivity patterns created by 7 point sources (illustrated with red circles) lobe number increases with frequency!



Fig. 2. Simulated far-field polar response of a 7-pointsource uniform array.



Radiator Groups Sound Pressure Directivity Patterns Wavefield Synthesis

Generally, the radiation patterns of group sources can be varied by varying the phases, amplitudes, and delays of the individual sources.



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The same principle can be used also for microphone arrays!



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The same principle can be used also for microphone arrays!

acoustic beamforming by summing the mic signals

