

Name \_\_\_\_\_ Student number \_\_\_\_\_

## Home assignment 1

Vibration of a torsion bar is described by the second order ordinary differential equations

$$\frac{GI_{rr}}{L} \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \begin{Bmatrix} \theta_{X2} \\ \theta_{X3} \end{Bmatrix} + \frac{\rho I_{rr} L}{6} \begin{bmatrix} 4 & 1 \\ 1 & 4 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_{X2} \\ \ddot{\theta}_{X3} \end{Bmatrix} = 0$$

in which  $I_{rr}$  is the second moment of area with respect to the axis of the bar,  $G$  is the shear modulus, and  $\rho$  is the density of material. Derive the angular speeds and the corresponding modes of the free vibrations.

### Solution template

The set of ordinary differential equations as given by the principle of virtual work  $\mathbf{M}\ddot{\mathbf{a}} + \mathbf{K}\mathbf{a} = 0$  consists of the inertia and stiffness parts. The symmetric mass matrix  $\mathbf{M}$  and the stiffness matrix  $\mathbf{K}$  depend on the structure. Angular speeds of the free vibrations are the eigenvalues of  $\mathbf{\Omega} = \sqrt{\mathbf{M}^{-1}\mathbf{K}}$ . In practice, it is easier to calculate first the eigenvalues  $\mathbf{\Omega}^2 = \mathbf{M}^{-1}\mathbf{K}$  as the eigenvalues of  $\mathbf{\Omega}$  are the square roots of those for  $\mathbf{\Omega}^2$  and the eigenvectors coincide.

In the present case, the matrices are

$$\mathbf{K} = \frac{GI_{rr}}{L} \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \quad \text{and} \quad \mathbf{M} = \frac{\rho I_{rr} L}{6} \begin{bmatrix} 4 & 1 \\ 1 & 4 \end{bmatrix} \quad \Leftrightarrow \quad \mathbf{M}^{-1} = \frac{1}{\rho I_{rr} L} \begin{bmatrix} 8/5 & -2/5 \\ -2/5 & 8/5 \end{bmatrix}.$$

Therefore

$$\mathbf{\Omega}^2 = \mathbf{M}^{-1}\mathbf{K} = \frac{G}{\rho L^2} \begin{bmatrix} 18/5 & -12/5 \\ -12/5 & 18/5 \end{bmatrix}.$$

In the eigenvalue problem of matrix  $\mathbf{A}$ , the goal is to find all pairs  $(\lambda, \mathbf{x})$  such that  $(\mathbf{A} - \lambda\mathbf{I})\mathbf{x} = 0$ . The linear homogeneous equation system can have a non-zero solution only if  $\det(\mathbf{A} - \lambda\mathbf{I}) = 0$ . The eigenvalues are obtained as solutions to this characteristic equation. The characteristic equation for the eigenvalues of  $\mathbf{\Omega}^2$  is

$$\det(\mathbf{\Omega}^2 - \lambda\mathbf{I}) = \left(\frac{18}{5} \frac{G}{\rho h^2} - \lambda\right)^2 - \left(\frac{12}{5} \frac{G}{\rho h^2}\right)^2 = 0.$$

The two solutions for the eigenvalues are  $((a - \lambda)^2 - b^2 = 0 \Leftrightarrow \lambda = a \pm b)$

$$\lambda_1 = \frac{6}{5} \frac{G}{\rho L^2} \quad \text{and} \quad \lambda_2 = 6 \frac{G}{\rho L^2}.$$

The corresponding eigenvectors are obtained as solutions to  $(\mathbf{A} - \lambda \mathbf{I})\mathbf{x} = 0$ . The eigenvectors are not unique and it is enough to find some of them. However, the eigenvectors should be linearly independent so that, e.g., the zero vector is not a valid choice.

$$\lambda_1: \quad \frac{G}{\rho L^2} \begin{bmatrix} 12/5 & -12/5 \\ -12/5 & 12/5 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = 0 \quad \Rightarrow \quad \mathbf{x}_1 = \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 1 \\ 1 \end{Bmatrix},$$

$$\lambda_2: \quad \frac{G}{\rho L^2} \begin{bmatrix} -12/5 & -12/5 \\ -12/5 & -12/5 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = 0 \quad \Rightarrow \quad \mathbf{x}_2 = \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 1 \\ -1 \end{Bmatrix}.$$

The representation of the matrix in terms of its eigenvalues and eigenvectors  $\mathbf{\Omega}^2 = \mathbf{X}\mathbf{\Lambda}\mathbf{X}^{-1}$  implies that  $\mathbf{\Omega} = \mathbf{X}\sqrt{\mathbf{\Lambda}}\mathbf{X}^{-1}$ . As taking a square root of the diagonal matrix means just taking the square roots of the diagonal terms, the angular speeds of the free vibrations

$$\omega_1 = \sqrt{\lambda_1} = \sqrt{\frac{6}{5} \frac{G}{\rho L^2}} \quad \text{and} \quad \omega_2 = \sqrt{\lambda_2} = \sqrt{6 \frac{G}{\rho L^2}}. \quad \leftarrow$$