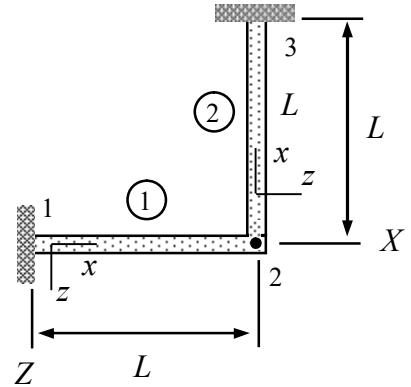


Name \_\_\_\_\_ Student number \_\_\_\_\_

## Home assignment 2

The  $XZ$ -plane structure shown consists of two beams of equal properties. Assuming that the beams are inextensible in the axial directions, derive the angular speed of free vibrations. Young's modulus  $E$  and density  $\rho$  of the beam material and the second moment of area  $I$  are constants. Assume that the inertia term due to rotation is negligible.



### Solution template

As the beams are inextensible in the axial directions, the active degree of the structure is  $\theta_{Y2}$ . Let us start with the element contributions. Only the virtual work expressions of the internal and inertia forces (available in the formulae collection) are needed.

$$\delta W^{\text{int}} = - \begin{Bmatrix} \delta u_{z1} \\ \delta \theta_{y1} \\ \delta u_{z2} \\ \delta \theta_{y2} \end{Bmatrix}^T \frac{EI_{yy}}{h^3} \begin{bmatrix} 12 & -6h & -12 & -6h \\ -6h & 4h^2 & 6h & 2h^2 \\ -12 & 6h & 12 & 6h \\ -6h & 2h^2 & 6h & 4h^2 \end{bmatrix} \begin{Bmatrix} u_{z1} \\ \theta_{y1} \\ u_{z2} \\ \theta_{y2} \end{Bmatrix},$$

$$\delta W^{\text{ine}} = - \begin{Bmatrix} \delta u_{z1} \\ \delta \theta_{y1} \\ \delta u_{z2} \\ \delta \theta_{y2} \end{Bmatrix}^T \left( \frac{\rho I_{yy}}{30h} \begin{bmatrix} 36 & -3h & -36 & -3h \\ -3h & 4h^2 & 3h & -h^2 \\ -36 & 3h & 36 & 3h \\ -3h & -h^2 & 3h & 4h^2 \end{bmatrix} + \frac{\rho Ah}{420} \begin{bmatrix} 156 & -22h & 54 & 13h \\ -22h & 4h^2 & -13h & -3h^2 \\ 54 & -13h & 156 & 22h \\ 13h & -3h^2 & 22h & 4h^2 \end{bmatrix} \right) \begin{Bmatrix} \ddot{u}_{z1} \\ \ddot{\theta}_{y1} \\ \ddot{u}_{z2} \\ \ddot{\theta}_{y2} \end{Bmatrix},$$

Virtual work expressions of internal and inertia forces for beam 1 simplify to

$$\delta W^{\text{int}} = - \begin{Bmatrix} 0 \\ 0 \\ 0 \\ \delta \theta_{Y2} \end{Bmatrix}^T \frac{EI}{L^3} \begin{bmatrix} 12 & -6L & -12 & -6L \\ -6L & 4L^2 & 6L & 2L^2 \\ -12 & 6L & 12 & 6L \\ -6L & 2L^2 & 6L & 4L^2 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ 0 \\ \theta_{Y2} \end{Bmatrix} = -\delta \theta_{Y2} 4 \frac{EI}{L} \theta_{Y2},$$

$$\delta W^{\text{ine}} = - \begin{Bmatrix} 0 \\ 0 \\ 0 \\ \delta \theta_{Y2} \end{Bmatrix}^T \frac{\rho AL}{420} \begin{bmatrix} 156 & -22L & 54 & 13L \\ -22L & 4L^2 & -13L & -3L^2 \\ 54 & -13L & 156 & 22L \\ 13L & -3L^2 & 22L & 4L^2 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ 0 \\ \dot{\theta}_{Y2} \end{Bmatrix} = -\delta \theta_{Y2} \frac{\rho AL^3}{105} \dot{\theta}_{Y2},$$

giving

$$\delta W^1 = -\delta\theta_{Y2} \left( 4 \frac{EI}{L} \theta_{Y2} + \frac{\rho AL^3}{105} \ddot{\theta}_{Y2} \right).$$

Virtual work expressions of internal and inertia forces for beam 2 simplify to

$$\delta W^{\text{int}} = - \begin{Bmatrix} 0 \\ \delta\theta_{Y2} \\ 0 \\ 0 \end{Bmatrix}^T \frac{EI}{L^3} \begin{bmatrix} 12 & -6L & -12 & -6L \\ -6L & 4L^2 & 6L & 2L^2 \\ -12 & 6L & 12 & 6L \\ -6L & 2L^2 & 6L & 4L^2 \end{bmatrix} \begin{Bmatrix} 0 \\ \theta_{Y2} \\ 0 \\ 0 \end{Bmatrix} = -\delta\theta_{Y2} 4 \frac{EI}{L} \theta_{Y2},$$

$$\delta W^{\text{ine}} = - \begin{Bmatrix} 0 \\ \delta\theta_{Y2} \\ 0 \\ 0 \end{Bmatrix}^T \frac{\rho AL}{420} \begin{bmatrix} 156 & -22L & 54 & 13L \\ -22L & 4L^2 & -13L & -3L^2 \\ 54 & -13L & 156 & 22L \\ 13L & -3L^2 & 22L & 4L^2 \end{bmatrix} \begin{Bmatrix} 0 \\ \ddot{\theta}_{Y2} \\ 0 \\ 0 \end{Bmatrix} = -\delta\theta_{Y2} \frac{\rho AL^3}{105} \ddot{\theta}_{Y2},$$

giving

$$\delta W^2 = -\delta\theta_{Y2} \left( 4 \frac{EI}{L} \theta_{Y2} + \frac{\rho AL^3}{105} \ddot{\theta}_{Y2} \right).$$

Virtual work expression of structure is the sum of element contributions.

$$\delta W = \delta W^1 + \delta W^2 = -\delta\theta_{Y2} \left( 8 \frac{EI}{L} \theta_{Y2} + 2 \frac{\rho AL^3}{105} \ddot{\theta}_{Y2} \right).$$

Finally, principle of virtual work and the fundamental lemma of variation calculus implies the ordinary differential equation of the form  $\ddot{\theta}_{Y2} + \omega^2 \theta_{Y2} = 0$

$$\ddot{\theta}_{Y2} + 420 \frac{EI}{\rho AL^4} \theta_{Y2} = 0.$$

Therefore, the angular speed of the free vibrations  $\omega$  is given by

$$\omega = \frac{2}{L^2} \sqrt{105 \frac{EI}{\rho A}}.$$
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