

# Fuel Injection Systems in Diesel and SI Engines

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#### **Motivation**

- Why learn about fuel injection systems?
- Fuel injection and fuel injection systems play a fundamental role in defining the engine combustion, fuel consumption, and emissions
  - When
  - Dosing
  - Direction
  - Mixing



# Fuel Injection Systems in Diesel Engines



## The goals of fuel injection

- Get the correct amount of fuel to the combustion chamber at the right time and to the right place
- Evaporate the liquid fuel
- Mix the fuel vapor with oxygen to obtain good ignitable mixture

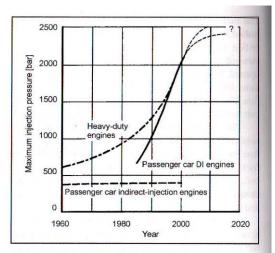
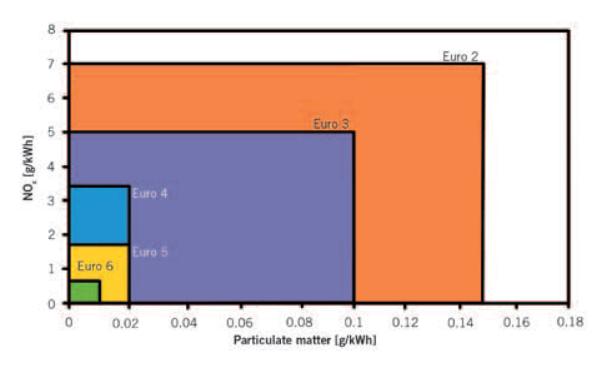


Fig. 12-21 Development of the maximum injection pressure in recent decades.

 In case the fuel injection is well achieved by the above criteria, the combustion process typically yields low fuel consumption, emissions and noice (→ no pressure peaks)



#### **Emission regulations**

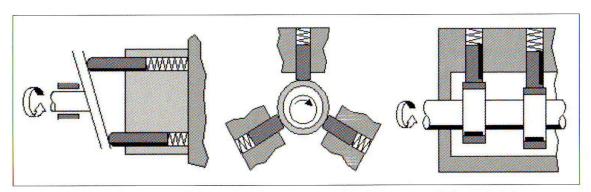


**EU PM-** and **NO**x-emission limits for cars with diesel engine

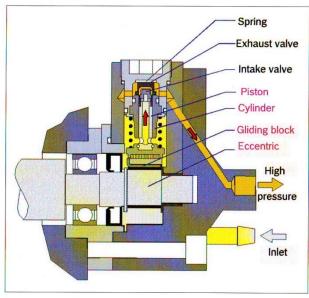
**Euro 6 will come into force 2015** 



## **High pressure pump**

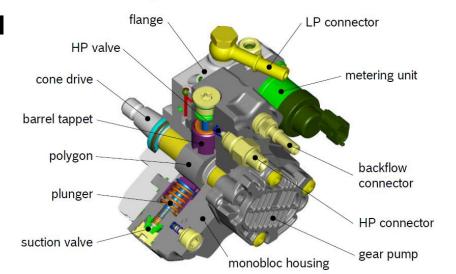


**Figure 4.7**: Basic designs of high pressure fuel pumps: axial piston pump (left); radial piston pump (center); in-line pump (right)



#### Common Rail High Pressure Pump (RB CP3.2)

- Volume flow ~ rotational speed
- Maximum pressure obtained already with relatively low rotational speed
- Control applied for the suction side

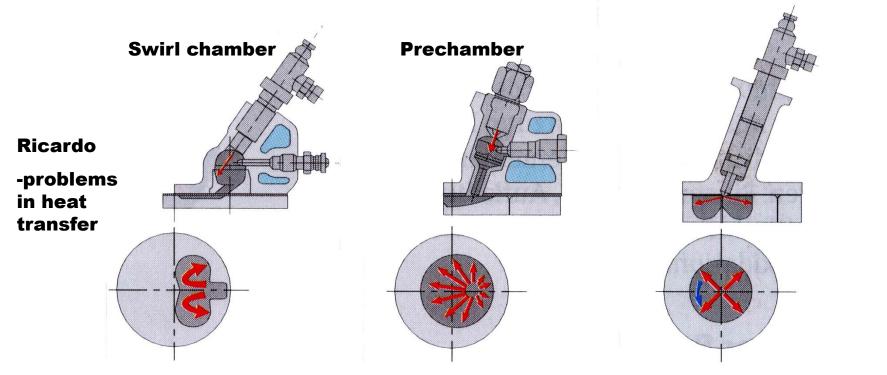




## Different combustion chamber types

#### **Indirect fuel injection**

#### **Direct fuel injection**



Pictures: MTZ Sonderausgabe 10 Jahre TDI-Motor von Audi



## Injection systems in Diesel engines

- Distributor systems
  - Axial piston distributor pump
  - Radial piston pump
- Unit pump systems
  - Pump units assembled in one unit: inline fuel injection pump
  - Separate pump units for every cylinder
  - United injector pump and nozzle, unit injector system and unit pump system
- Common Rail (accumulator) systems



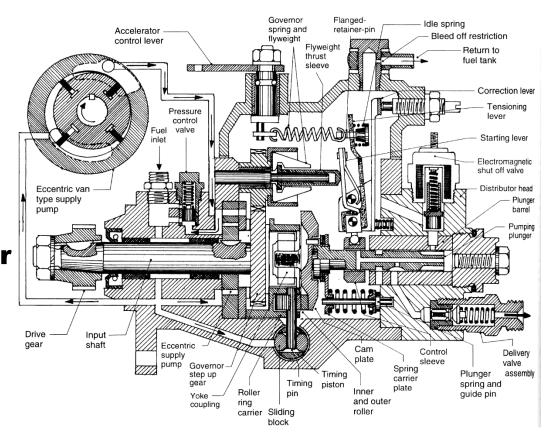
#### **Development**

- 1927 Inline pump
- 1930 Diesel fuel filter
- 1962 Distributor pump
- 1986 EDC (Electronic Diesel Control)
- 1994 UIS (unit injector system)
- 1995 UPS (unit pump system)
- 1996 VR-radial piston pump (VP44)
- 1997 CRS (Common Rail System)
- 2001 CRS 2. generation
- 2003 CRS 3. generation (pietzo injector)



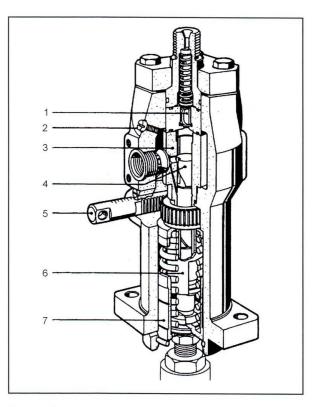
### **Distributor pump**

- Only one pump unit for all cylinders
- Nowadays is more and more replaced by CR systems
- Previously used together with pre- and swirl chamber injection in all cars and boats





## Inline pump (unit pump)



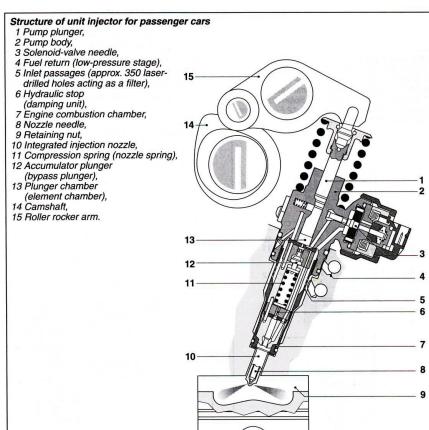
**Fig. 12-29** Mechanically controlled single-plunger fuel injection, individual injection pump for large engines<sup>6</sup>: 1, Pressure valve; 2, Vent screw; 3, Pump cylinder; 4, Pump piston; 5, Control rack; 6, Control sleeve; 7, Guide bushing.

- Separate pump unit for each cylinder
- Tractors and on the other hand power plant / ship engines
- In power plants called Unit pumps (although the principles are the same)



## Unit Injector System (UIS)

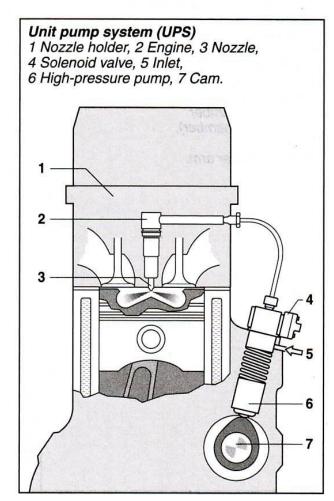
- High-pressure pump and nozzle are connected. No high-pressure pipe.
- Small volume to be pressurized
   → highest maximum pressures
- Used mainly in cars





## Unit Pump System (UPS)

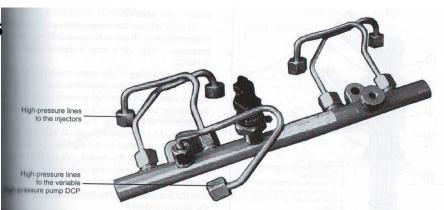
 Difference to the UIS system is the high-pressure pipe that allows more flexible location of the pump unit.





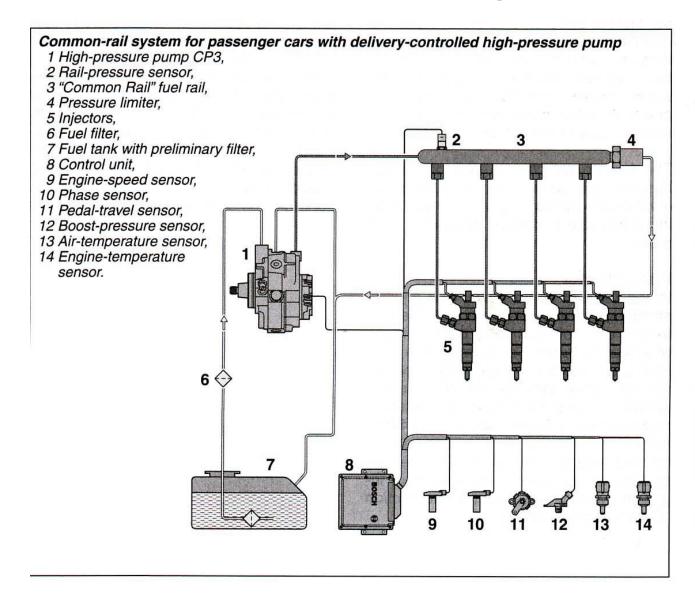
## **Common Rail System**

- Common pressure reservoir into which all the injector nozzles are connected. Pressure pipe is connected to the injector solenoid valve
- Typical injection pressure level 1200...2000 bar
- Pressure level can be freely chosen
- The start and ending of the fuel injection are independent of the cam shaft
- Pre- and post injections
- Used in cars and tractors
- More and more used in power plants and ships





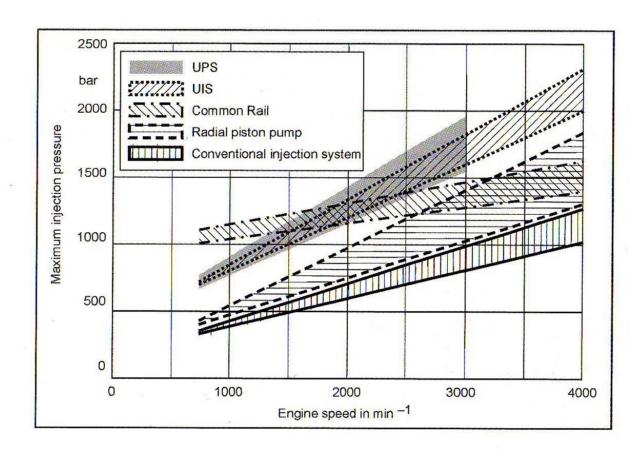
## **Common Rail System**





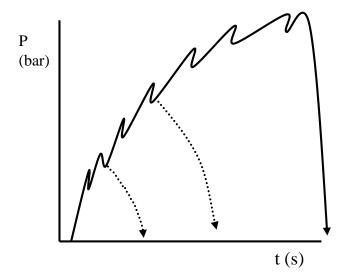
#### **Pressure levels**

- UPS = Unit Pump System
- UIS = Unit Injector System (or Pump Nozzle Unit)

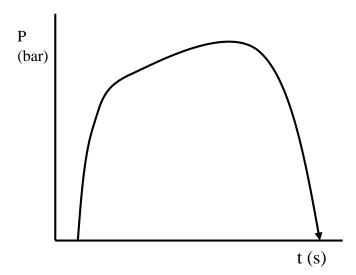




## **Comparing injection pressures**



Traditional injection pressure (inline- or distributor pumps)



**Common Rail injection** 



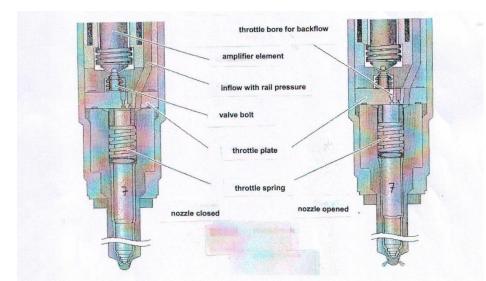
#### **Injectors and Nozzles**

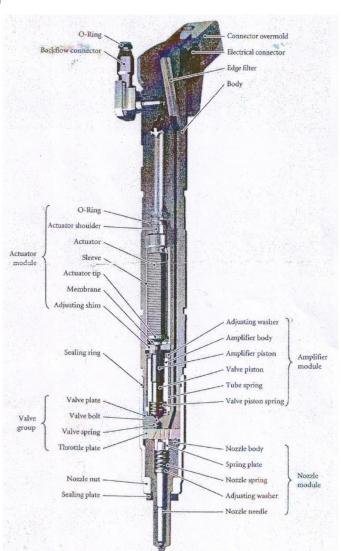
- The goal of the injector and nozzle is to inject the pressurized fuel so that fast droplet breakup and evaporation are achieved, and hence efficient combustion is achieved
- Terminology:
  - Injector = takes in high-pressure fuel. May be solenoid controlled. Nozzle is attached to the injector. Delivers fuel to the nozzle
  - Nozzle = is attached to injector. Distributes fuel into the combustion chamber. Has typically many holes for fuel exit
  - In Finnish:
    - Injector = suutinrunko, injektori
    - Nozzle = suutin, suutinkärki



## **Working principle**

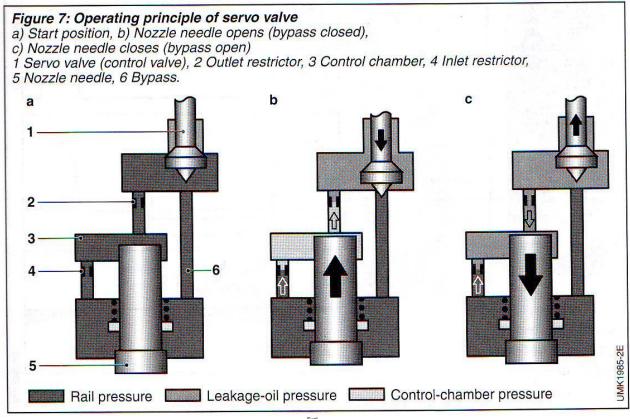
- Same pressure both on the top and bottom areas of the needle: needle does not move
- Injection starts: pressure decreases on top of the needle
- Needle raises because there is a pressure shoulder in the needle into which an upwards directed force is acting

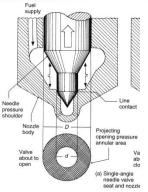






#### Working principle

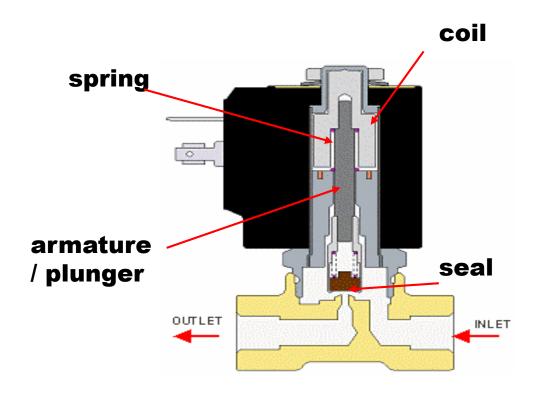


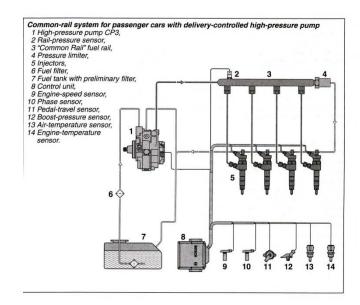




#### **Solenoid valve**

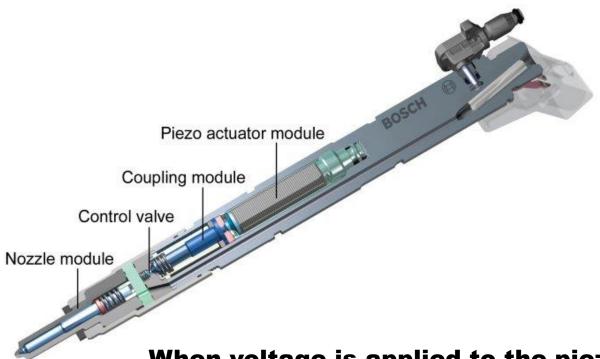
Used typically together with CR injection







## Piezo Injector valve

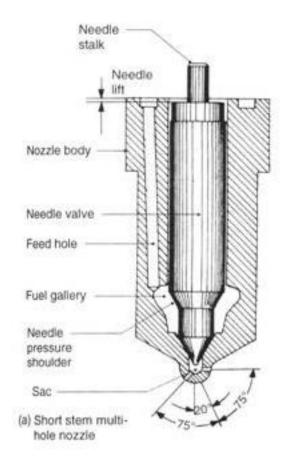


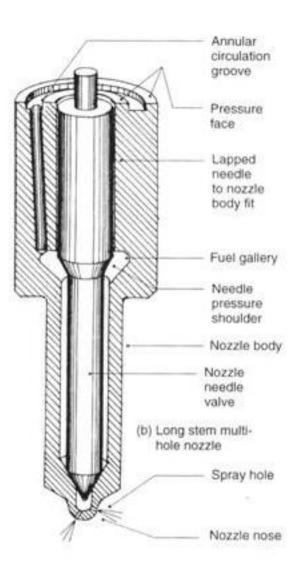
When voltage is applied to the piezo stack (actuator module), its size increases ( $\sim$ 60 $\mu$ m). Piezo injectors are very fast (<100 micro-s), light, and consume low amount of energy.



- Nozzle delivers fuel to the combustion chamber and participates to the breakup of fuel into droplets
- Has great role in successfull combustion process

#### **Nozzles**

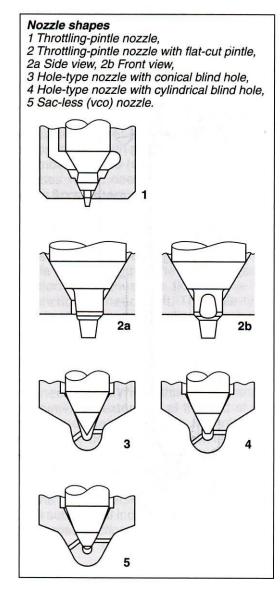






#### **Nozzles**

 Pintle nozzles are mainly used in direct injection SI engines



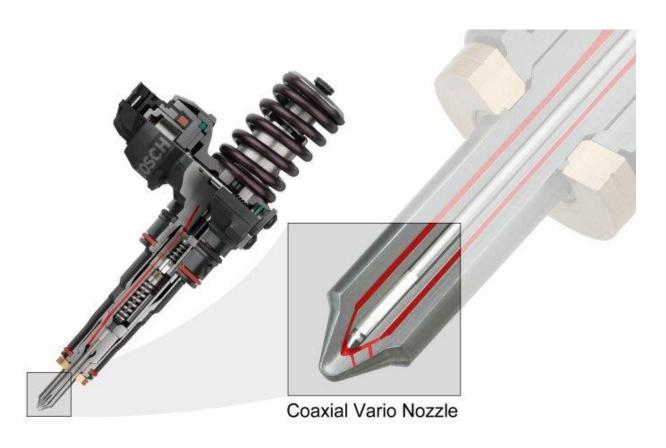


#### **Duel nozzle diesel & water**





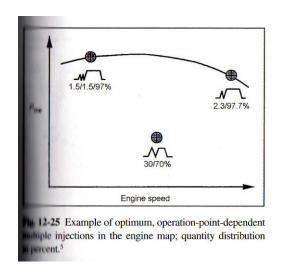
### **Bosch**

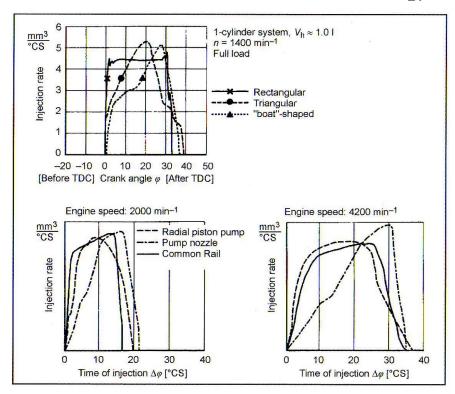


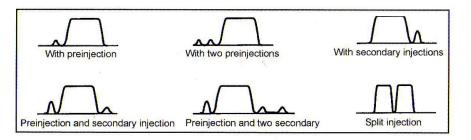


## Injection profiles

 Influences engine fuel consumption, power, emissions, and noice









## Calculating the theoretical fuel exit velocity form a nozzle

Theoretical fuel exit velocity from nozzle

$$p_1 + \frac{1}{2}\rho u_1^2 = p_2 + \frac{1}{2}\rho u_2^2 \longrightarrow U_{Exit} = \sqrt{\frac{2\Delta P}{\rho_f}}$$

Velocity in practice

$$U_{Exit} = C_v \sqrt{\frac{2\Delta P}{\rho_f}}$$

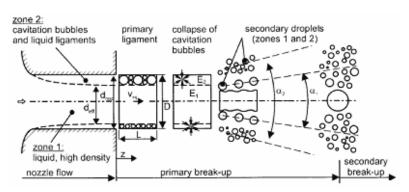


### Discharge coefficient Cd

Discharge coefficient

$$C_d = C_v \cdot C_a$$

- $C_d$  used when calculating mass flow  $\dot{m} = C_d A \rho_f \sqrt{\frac{2\Delta P}{\rho_f}}$
- $C_{_{\scriptscriptstyle \mathcal{V}}}$  'velocity coefficient', used when calculating real injection velocity
- $C_a$  'area contraction coefficient', used when calculating effective nozzle hole diameter





#### **Pumping power**

- The power needed to pressurize fuel
- Isentropic pumping power

$$P_s = \dot{V}(P_2 - P_1)$$

Practical pumping power

$$P_{tod} = rac{P_{_S}}{\eta_{_S}}$$
  $\eta_{_S}$  = isentropic efficiency

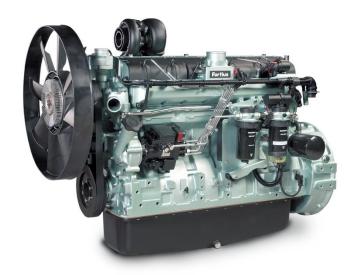


## Comparing car and tractor diesel engines

BMW 123d

Sisudiesel 66 ETA







## **Engine comparison**

		BMW 123d	Sisudiesel 66 ETA
$\Longrightarrow$	Teho	150 kW / 4000 r/min	150 kW / 2200 r/min
$\Longrightarrow$	Vääntömomentti	400 Nm / 2000 - 2250 r/min	850 Nm / 1200 - 1500 r/min
	Sylinteriluku	4	6
	Iskun pituus	90,0 mm	120 mm
$\Longrightarrow$	Sylinterin halkaisija	84,0 mm	108 mm
	Iskutilavuus	1998 cm <sup>3</sup>	6600 cm <sup>3</sup>
	Ominaisteho	75.2 kW/l	22.7
	Minimi ominaiskulutus	204 g/kWh	198 g/kWh
	Puristussuhde	16,0	16,5
$\Longrightarrow$	Massa	161 kg	510 kg
	Mitat (pit. $\times$ lev. $\times$ kork.)	$717\times842\times688~mm$	1185 × 645 × 997 mm



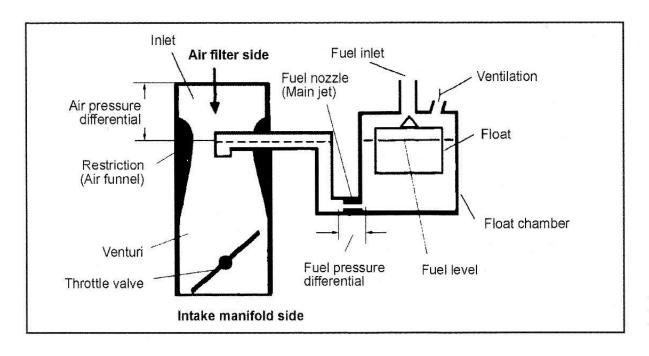
# Fuel Injection Systems in Gasoline Engines

Kul-14.4700 Transport Biofuels, Combustion, and Emission Control
2014

Ossi Kaario



#### **Carburetor**



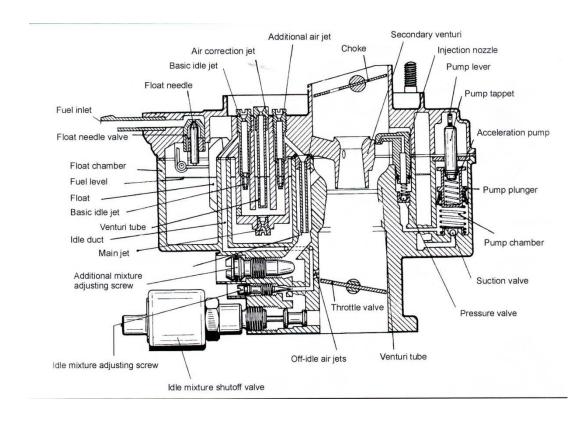
**Fig. 12-1** Functional principle of the carburetor.

**Bernoulli** 
$$p_1 + \frac{1}{2}\rho u_1^2 = p_2 + \frac{1}{2}\rho u_2^2$$



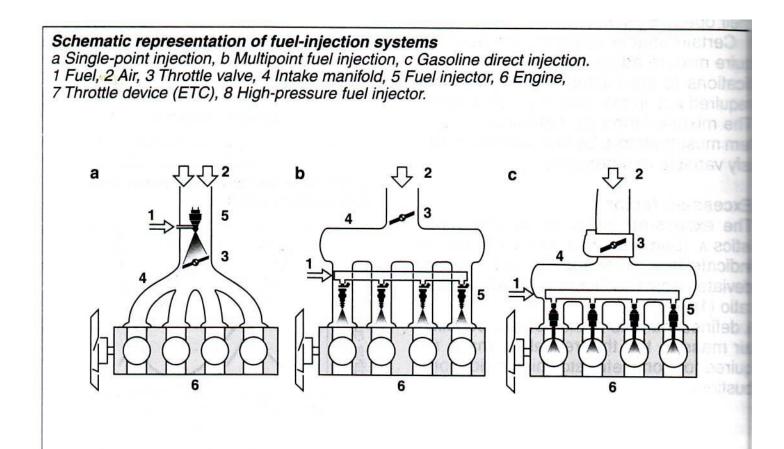
- Old but well working technology
- Problems with e.g. cold starting (fuel vapor condensation to cold surfaces) and accurate fuel delivery
- Utilized in e.g. chain saws or lawn movers
- Secondary venturi

#### **Carburetor**





### **Fuel injection**

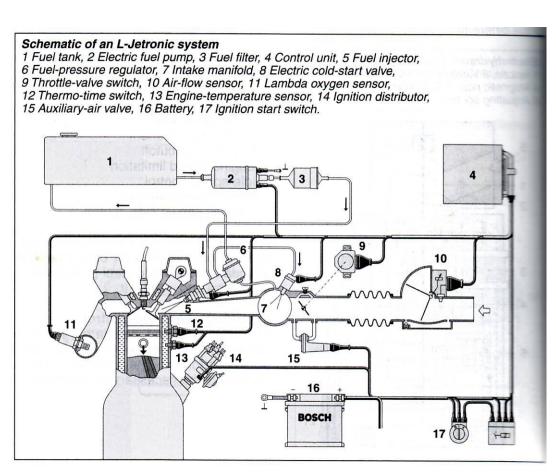




### **Multi-point injection**

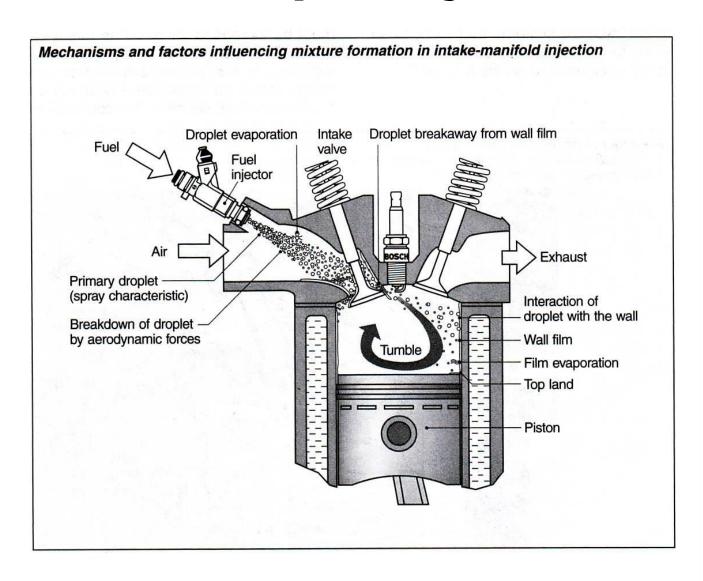
- Injection pressure3...5bar
- Controls:

Starting enrichment
Acceleration control
Full throttle controll
Fuel cut-off
Limit to rotational speed
Idle control
Altitude control
Lambda control



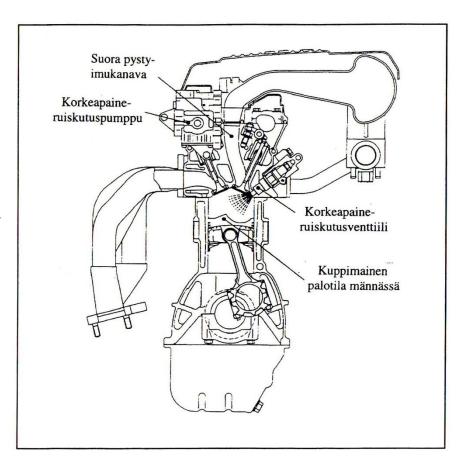


### **Multi-point injection**





- Reduced fuel consumption close to 20%
- Part load stratified charge → Full load homogeneous charge
- Control demanding
- Not as sensitive to knocking behavior compared to intake manifold injection systems



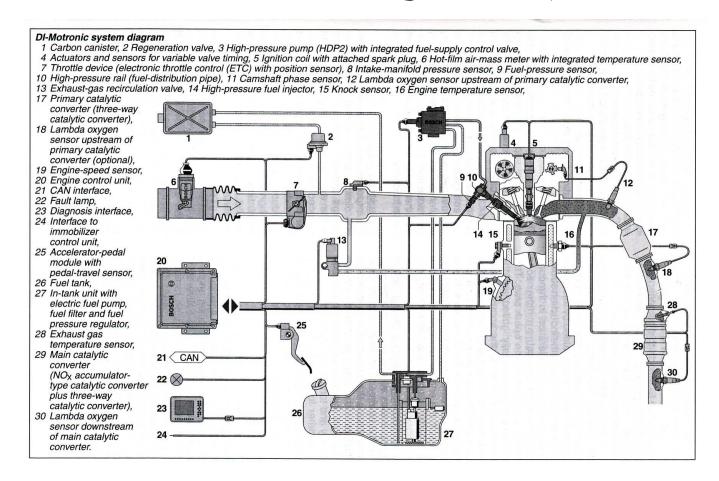
Kuva 2.4. Nelisylinterisen Mitsubishi 4G93-suoraruiskutusbensiinimoottorin poikkileikkaus. Sylinterin halkaisija D = 81 mm, iskunpituus S = 89 mm, puristussuhde e = 12, palotila osittain sylinterin kannessa, osittain männässä, lähes pystysuora imukanava, kaksi yläpuolista nokka-akselia /2.4/.





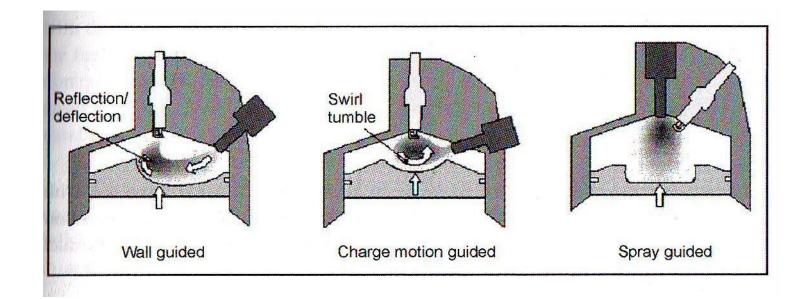


#### Gasoline direct injection, Bosch



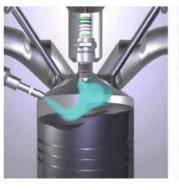
Injection pressures 50...200 bar











Kerrossyöttö

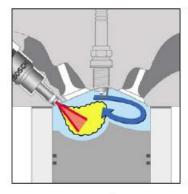
- $\lambda = 1...3$
- · Kerroksittainen seos



Homogeeninen

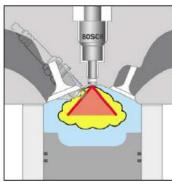
- $\cdot \lambda = 1$
- Tasalaatuinen seos

#### Täytöstavat:



Täytöstä ohjataan sylinteritilan muotoilun avulla

Seinämäohjattu

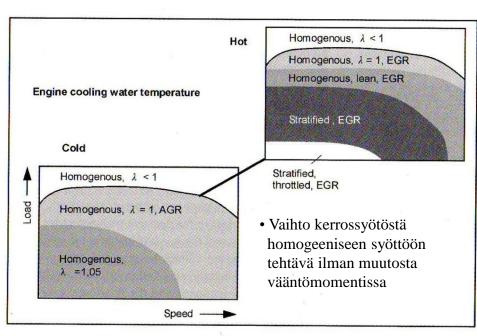


Suihkutusohjattu Täytöstä ohjataan polttoainesuihkun avulla

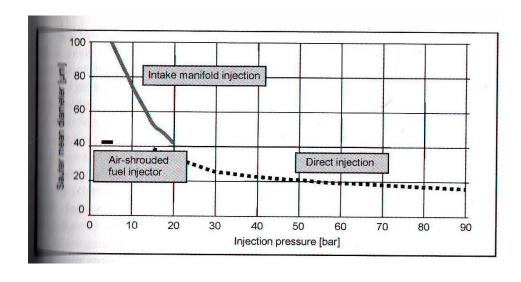


# Direct fuel injection, mixture formation

- High load homogeneous charge, total Lambda close to 1
- Partial load stratified charge, high total Lambda
- Combustion chamber has two charge areas:
  - Close to spark goal is Lambda=1
  - Elsewhere "insulation"
     layer between ignitable
     mixture and cylinder
     wall (air + egr)







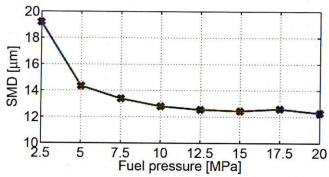


Figure 13. Influence of fuel pressure on the Sauter mean diameter (D32)of droplets in the spray core. Measurements taken in a heated spray chamber under evaporating conditions (1 MPa/600 K) [78].

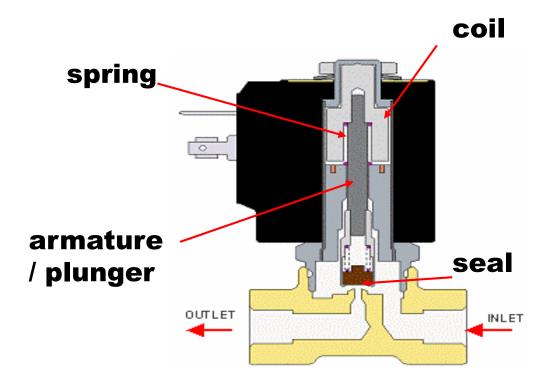


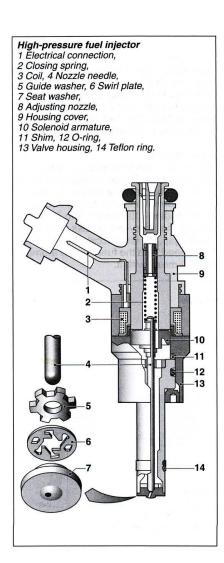
# Why does much lower fuel injection pressure suffice in SI engines compared to diesel engines?



#### **Solenoid valves**

- Control: electric current in the coil produces a magnetic field which raises the plunger
- Direct or indirect acting







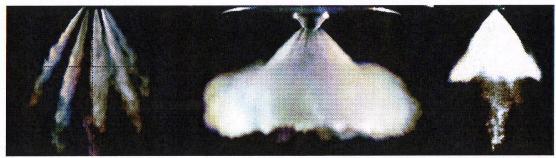
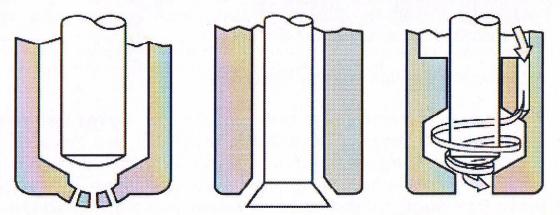


Figure 15. Examples of sprays from three types of injectors: left, multi-hole; center, outwards opening A-nozzle; right, an inwards opening swirl type nozzle. Note that the images are not taken using the same geometrical scale.



M. Skogsberg, Phd thesis, 2007, Chalmers

Figure 16. Cross-sections of three types of injector; left, multi-hole; center, A-nozzle; right, inwards opening swirl type nozzle. Reprinted with permission from SAE Paper No. 980498 [30], © 1998 SAE International.



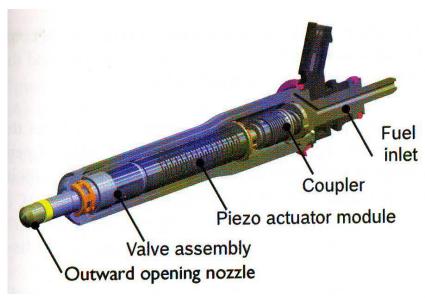
	Swirl	Outward Opening	Multi-Hole
Spray Stability / Tolerance	+	++	+
Flexibility of Spray Pattern	+	0	++
Resistance against backpressure influence	,	++	++
Multi-Injection capability	0	+	0
Costs	0	_*	+
Robustness against plugging	+	+	+

<sup>\*)</sup> in combination with piezo-electric actuation

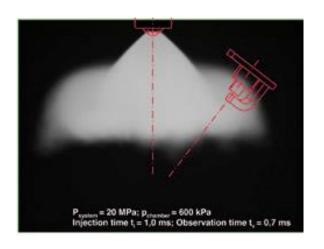
Figure 18. Atomization concepts for DISC fuel injectors. Reprinted with permission from SAE Paper No. 2007-01-1404 [7], © 2007 SAE International.



### Injector for direct injection



- Piezo element replaces solenoid valve
- Fast
- Low energy consumption

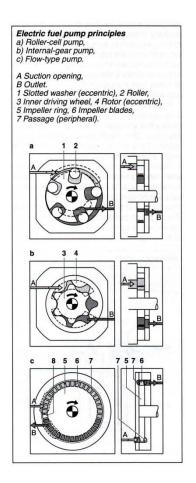


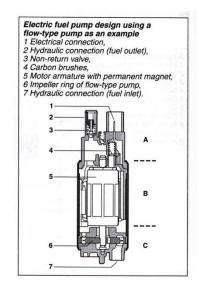




#### **Pressurizing fuel**

#### Low pressure systems





#### **High pressure systems**

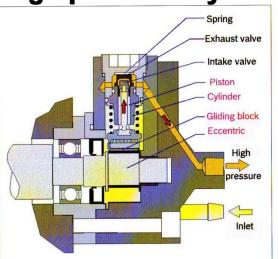
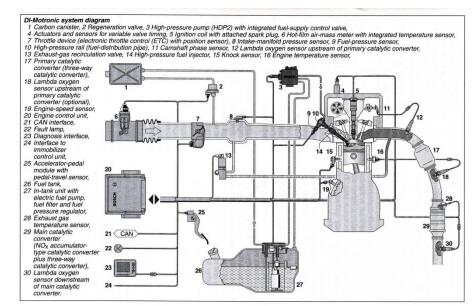
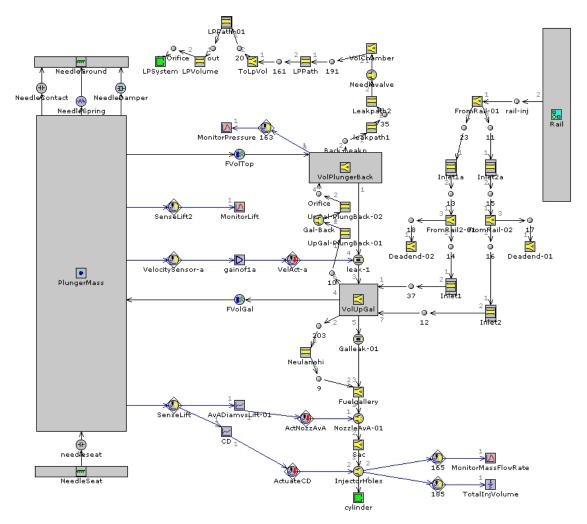


Figure 4.8:
Three-cylinder radial piston pump
[18]





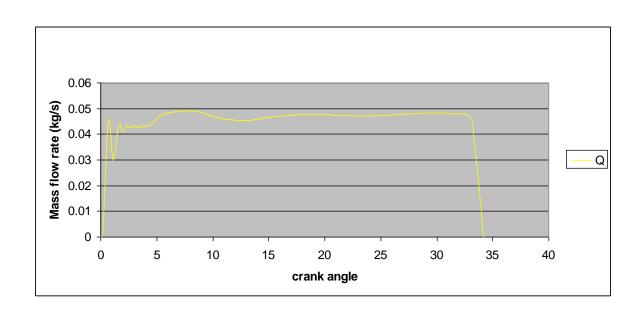
#### **Simulation**



# GT-Fuel model of EVE injection system



#### **Simulation**



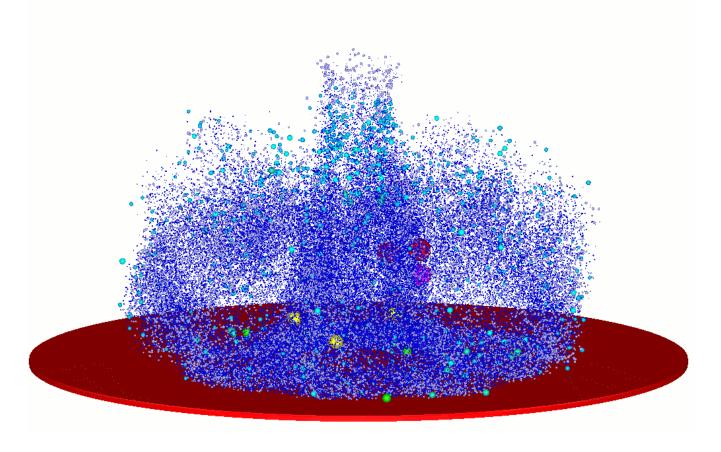
## **Computed mass flow rate in the nozzle hole**

Fig. A. Tilli

More about 1-D simulations, course Kul-14.4400



#### **CFD Simulation**



HCCI type fuel sprays obtained by CFD Fig. O. Kaario

More about CFD simulations, course Kul-14.5200