

Homework 2

CIV-E4080, Material Modeling in Civil Engineering L

Introduction

Recall of a necessary minimum of some basic concepts and equations of solid mechanics necessary to follow this course.

Readings

1. Chapter 1 from: **D. Gross and T. Seelig**, *Fracture Mechanics: With an Introduction to Micromechanics*.
2. **Reddy**; Chapters 3 (*kinematics of continua*) and 4 (*stress measures*)
3. Also refer to the Appendix I added to MyCourses. *Recall: Stress invariants*.

Problem 1 - Stress Invariants

[5 points] Consider the plane stress state below,

$$\sigma = \begin{bmatrix} \sigma_o & 0 & 0 \\ 0 & \alpha\sigma_o & 0 \\ 0 & 0 & 0 \end{bmatrix} \quad (1)$$

where, $\alpha \in [-1, 1]$, $\sigma_o > 0$ and x_1, x_2, x_3 , coordinates in the basis $\vec{i}_1, \vec{j}_2, \vec{k}_3$.

Experimental setting giving rise to such bi-axial stress state is often used to determine parameters of material models.

Determine as a function of the parameter α ,

1. The mean stress α_m , called also the hydrostatic pressure - p.
2. The equivalent, von Mises, stress ($\sigma_e = \sqrt{3J_2}$).
3. Lode angle $\theta \equiv 1/3 \arccos(3\sqrt{3}/2 \cdot J_3/J_2^{3/2})$
4. The maximum shear stress τ_{max} .
5. The normal of the maximum shear stress plane

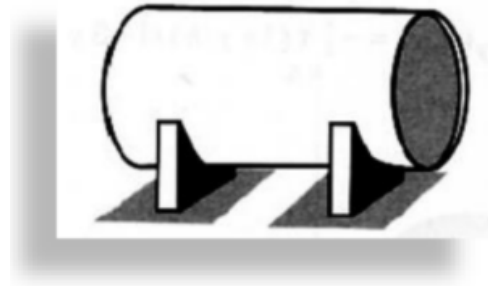
What is the stress state corresponding to $\alpha = -1$?

Solution

1. Hydrostatic Pressure / Mean Stress, $\alpha_m = (\sigma_o + \alpha\sigma_o + 0)/3 = \sigma(1 + \alpha)/3$
2. $J_2 = \frac{1}{6}[(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2] \Rightarrow \frac{1}{3}\sigma_o^2(1 - \alpha + \alpha^2)$
 $\sigma_e = \sqrt{3J_2} \Rightarrow \sqrt{(1 - \alpha + \alpha^2)}\sigma_o$
3. $J_3 = \det s \Rightarrow -\frac{1}{9}(2 - \alpha)(2\alpha - 1)(1 + \alpha)\sigma_o^3$
 $\theta \equiv 1/3 \arccos(3\sqrt{3}/2 \cdot J_3/J_2^{3/2})$
 $\Rightarrow 1/3 \arccos(3\sqrt{3}/2 \cdot (-\frac{1}{9}(2 - \alpha)(2\alpha - 1)(1 + \alpha)\sigma_o^3)/(\frac{1}{3}\sigma_o^2(1 - \alpha + \alpha^2))^{3/2})$
4. $\tau_{max} = \frac{1}{2}(\sigma_1 - \sigma_3) \Rightarrow \frac{1}{2}(\sigma_o)$
5. The maximum shear stresses are in planes forming angles of 45 degrees with Principal Planes.

Problem 2 - Pressure Vessel

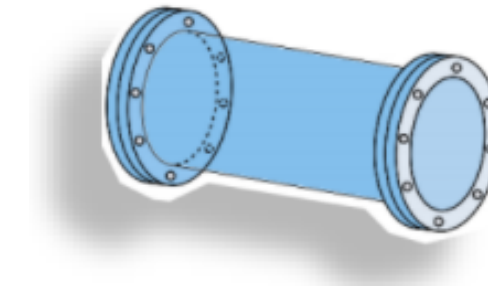
[5 points] Consider a thin-walled metallic cylindrical pressure vessel shown in the figure below.



$t/R \ll 1$ – very thin

$D/2 = R$ – radius

t – thickness $\alpha > 0$,



The pressure vessel (the thin walled tube) is loaded with an internal pressure p_o and a torsion moment $M_t = \alpha p_o t D^2$ resulting in a bi-axial membrane stress state. Assume a membrane stress state for the tube with a cylindrical symmetry.

1. Determine the stress state in the thin-wall of the cylinder (membrane stress state with a cylindrical symmetry).
2. Plot the stress state in the deviatoric plane for $\alpha > \dots$ i.e., plot a 'stress point' $P(\sigma)$ using the coordinates (ρ, θ)

Hints

$J_2 = \frac{1}{2} s : s$, $J_3 = \det s$, Deviatoric radius: $\rho = \sqrt{2J_2}$ and Lode angle: $\cos 3\theta = \left(\frac{3\sqrt{3}}{2} \frac{J_3}{J_2^{3/2}}\right)$.

Solution

1. Since, the thickness to diameter ratio is very small, $t/D \ll 1$, we can assume to have uniform stress distribution in the shell and a plane stress state. The respective stress equations are,

$$\sigma_x = \frac{1}{2}p_o \frac{D}{t}, \sigma_y = \frac{1}{4}p_o \frac{D}{t}, \tau_{xy} = M_x/W_v = \alpha p_o t D^2 / \frac{1}{4}\pi D^2 t = (4\alpha/\pi)p_o$$

The Stress Matrix is,

$$\sigma = \begin{bmatrix} \frac{1}{2} \frac{D}{t} & (4\alpha/\pi) & 0 \\ (4\alpha/\pi) & \frac{1}{4} \frac{D}{t} & 0 \\ 0 & 0 & 0 \end{bmatrix} p_o$$

$$\sigma = \begin{bmatrix} \frac{1}{2} & \beta & 0 \\ \beta & \frac{1}{4} & 0 \\ 0 & 0 & 0 \end{bmatrix} p_o D/t$$

Where, $\beta = 4\alpha t/\pi D$

2. The mean stress is,

$$\sigma_m = \frac{1}{3}tr\sigma \Rightarrow (1/4)p_o D/t$$

The deviatoric stress becomes,

$$s = \begin{bmatrix} \frac{1}{4} & \beta & 0 \\ \beta & 0 & 0 \\ 0 & 0 & -\frac{1}{4} \end{bmatrix} p_o D/t$$

$$J_2 = \frac{1}{2}s_{ij}s_{ji} = \frac{1}{2}(2\frac{1}{16} + 2\beta^2)p_o^2 D^2/t^2 \Rightarrow (\frac{1}{16} + \beta^2)p_o^2 D^2/t^2$$

$$J_3 = det s \Rightarrow \frac{1}{4}\beta^2(p_o D/t)^3$$

$$\text{Deviatoric Radius, } \rho = \sqrt{2J_2} \Rightarrow \sqrt{\frac{1}{8} + 2\beta^2} p_o D/t$$

$$\text{Lode Angle, } \cos 3\theta = \frac{3\sqrt{3}}{2} \frac{J_3}{J_2^{3/2}} \Rightarrow \frac{3\sqrt{3}}{8} \frac{\beta^2}{(\frac{1}{16} + \beta^2)^{3/2}}$$

The stress point on deviatoric plane can be plotted by polar coordinates ρ and θ as a functions of β , which is directly proportional to α .