

#### Lecture 7: Laminar premixed flames

(related : Warnatz, Ch. 3, 5, 8, 12)

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## **Upcoming Sessions**

- Lecture 6: Laminar premixed flames
- Lecture 7: Details of laminar premixed flames
- Exercise 4 on theory and practice of laminar flame computations
- Exercise 5: Simple emission computations with Cantera
- My Course web interface for discussions and questions: https://mycourses.aalto.fi/



## **Premixed (PM) flames (Recap)**

- Reactants are perfectly premixed prior to ignition
  - $\rightarrow$  all components necessary for the reaction are present in the fuel
  - $\rightarrow$  to initiate reaction one has only to ignite the mixture
  - $\rightarrow$  complete combustion
- Formation of a propagation front
  - Front separates unburned from fully burned
  - self-sustaining
- Targeted to lean conditions
  - $\rightarrow$  complete combustion  $\rightarrow$  "no" soot  $\rightarrow$  no bright flame
  - $\rightarrow$  Visibility depends on fuel: e.g. blue glow of the premixed bunsen flame originates from excited states of CH, C2 (intermediate species in oxidization)
  - $\rightarrow$  Lower combustion temperatures  $\rightarrow$  lower Nox
- Applications : Gasoline and natural gas engines, modern gas turbines, explosions



### Chemical structure of PM flames (Recap)

Stoichiometric hydrogen flame



H<sub>2</sub>–Air,  $\phi$ =1.0, p=1 atm



## **Chemical structure of PM flames**





# Chemical structure of PM flames Heat release in stoichiometric hydrogen flame (Recap)





### Chemical structure of PM flames Important notes: (Recap)

- PM Flame has a finite thickness
- Mass and heat diffusion are very important
- Radicals and intermediate species can diffuse and initiate otherwise non-active reactions
- Reactions take place throughout the flame thickness
- Max heat release takes place at low T



### Laminar flame propagation

- Propagating flame front is an intrinsic feature of PM flames
- What influences to the propagation velocity ?
- How can we estimate this <u>"laminar flame speed" ?</u>



https://www.youtube.com/watch?v=IwjiVdk\_msA



https://upload.wikimedia.org/wikipedia/commons/e/e3/Blow\_ Torch\_(3257353199).jpg



### **Deflagration vs. detonation**

- PM flame propagation is a wave phenomena
  - Subsonic = deflagration
  - Supersonic = detonation
- Deflagration waves:
  - pressure and density decrease across the wave
  - Deflagration wave considered near isobaric
- Detonation
  - pressure and density increase across the wave



### **Deflagration vs. detonation**

- PM flame propagation is a wave phenomena
  - Subsonic = deflagration
  - Supersonic = detonation

Most important in practical applications

Deflagration waves:

- pressure and density decrease across the wave
- Deflagration wave considered near isobaric

Detonation

- pressure and density increase across the wave



• Conservation of mass

$$\frac{\partial \rho}{\partial t} + \operatorname{div}(\rho \vec{v}) = 0 . \qquad (12.7)$$

• Species  $\rho_i = w_i \rho$ 

$$\frac{\partial \rho_i}{\partial t} + \operatorname{div}(\rho_i \,\vec{v}) + \operatorname{div} \vec{j}_i = M_i \,\omega_i \,. \tag{12.8}$$

• Momentum

$$\frac{\partial(\rho\vec{v})}{\partial t} + \operatorname{div}(\rho\vec{v}\otimes\vec{v}) + \operatorname{div}\overline{p} = \rho\vec{g}. \qquad (12.9)$$

• Specific enthalpy

$$\frac{\partial(\rho h)}{\partial t} - \frac{\partial p}{\partial t} + \operatorname{div}(\rho \vec{v} h + \vec{j}_q) + \overline{p}: \operatorname{grad} \vec{v} - \operatorname{div}(p \vec{v}) = q_{\mathrm{r}}. \quad (12.12)$$



• Conservation of mass

$$\frac{\partial \rho}{\partial t} + \operatorname{div}(\rho \vec{v}) = 0. \qquad (12.7)$$
• Species  $\rho_i = w_i \vec{p}$  Typically expressed in mass fractions
$$\frac{\partial \rho_i}{\partial t} + \operatorname{div}(\rho_i \vec{v}) + \operatorname{div} \vec{j}_i = M_i \omega_i^{\bullet}. \qquad (12.8)$$
Species-wise diffusion  $\rho_i \vec{V}_i = \vec{j}_i$ 

• Momentum

$$\frac{\partial(\rho\vec{v})}{\partial t} + \operatorname{div}(\rho\vec{v}\otimes\vec{v}) + \operatorname{div}\overline{p} = \rho\vec{g}. \quad (12.9)$$
  
Remember, viscosity in pressure

Specific enthalpy  

$$\frac{\partial(\rho h)}{\partial t} - \frac{\partial p}{\partial t} + \operatorname{div}(\rho \vec{v} h + \vec{j}_q) + \vec{p}: \operatorname{grad} \vec{v} - \operatorname{div}(p \vec{v}) = q_r . \quad (12.12)$$
Aalto University  
School of Engineering
Heat flux
e.g. radiation

Driving force ⇒ ↓ Flux	Velocity gradient	Temperature gradient	Concentration gradient
Momentum	Newton's law $[\mu]$		
Energy		Fourier's law [λ]	Dufour effect $[D_i^T]$
Mass		Soret effect $[D_i^T]$	Fick's law [ <i>D</i> ]

**Fig. 5.8.** Fluxes and driving forces in transport processes (Onsager 1931, Hirschfelder et al. 1964)

- Heat flux consists of
- Heat conduction (c)
- Dufour effect (negligible) (D)
- Flux due to mass diffusion (d)

$$\frac{\partial(\rho h)}{\partial t} - \frac{\partial p}{\partial t} + \operatorname{div}(\rho \vec{v} h + \vec{j}_q) + \vec{p}: \operatorname{grad} \vec{v} - \operatorname{div}(p \vec{v}) = q_{\mathrm{r}}. \quad (12.12)$$
$$\vec{j}_q = \vec{j}_q^{\mathrm{c}} + \vec{j}_q^{\mathrm{D}} + \vec{j}_q^{\mathrm{d}},$$

$$\vec{j}_{q}^{c} = -\lambda \operatorname{grad} T$$

$$\vec{j}_{q}^{D} = \overline{M}RT \sum_{i} \sum_{j \neq i} \frac{D_{i}^{T}}{\rho D_{ij} M_{i} M_{j}} \left( \frac{w_{j}}{w_{i}} \vec{j}_{i} - \vec{j}_{j} \right)$$

$$\vec{j}_{q}^{d} = \sum_{i} h_{i} \vec{j}_{i}$$

$$\frac{Binary \ diffusion \ coefficient \ constant \ constan$$



- Diffusion flux consists of
- Diffusion due to concentration (d)
- Thermal diffusion **(T)**
- Pressure diffusion (<u>negligible</u>) (p)

$$\frac{\partial \rho_i}{\partial t} + \operatorname{div}(\rho_i \vec{v}) + \operatorname{div}_i \vec{j}_i = M_i \omega_i . \qquad (12.8)$$
$$\vec{j}_i = \vec{j}_i^{\mathrm{d}} + \vec{j}_i^{\mathrm{T}} + \vec{j}_i^{\mathrm{p}}$$

$$\vec{j}_{i}^{d} = \rho_{i}\vec{V}_{i} = \frac{\rho M_{i}}{\overline{M}^{2}}\sum_{j\neq i} D_{ij}^{\text{mult}} M_{j} \operatorname{grad} x_{j}$$

$$\vec{j}_{i}^{\mathrm{T}} = -D_{i}^{\mathrm{T}} \operatorname{grad}(\ln T)$$

$$\underline{Multicomponent}_{diffusion \ coefficient}$$

$$\vec{j}_{i}^{p} = \frac{\rho M_{i}}{\overline{M}^{2}}\sum_{j\neq i} D_{ij}^{\text{mult}} M_{j} (x_{j} - w_{j}) \operatorname{grad}(\ln p)$$



#### <u>Computing the multicomponent diffusion coefficient</u>

- From the kinetic theory of gases
- Very expensive
- Many theoretical and practical computational tricks needed
- Information (beyond course coverage) from Warnatz Ch. 5

$$\begin{split} L_{jk}^{00,00} &= \frac{16T}{25p} \sum_{l=1}^{K} \frac{X_l}{m_j \mathcal{D}_{jl}} \left\{ m_k X_k \left( 1 - \delta_{jl} \right) - m_j X_j \left( \delta_{jk} - \delta_{kl} \right) \right\}. \\ L_{jk}^{00,10} &= \frac{8T}{5p} \sum_{l=1}^{K} X_k X_l \left( \delta_{jk} - \delta_{jl} \right) \frac{m_l \left( 1.2C_{kl}^* - 1 \right)}{(m_k + m_l) \mathcal{D}_{kl}}. \\ L_{jk}^{10,00} &= L_{kj}^{00,10}. \\ L_{jk}^{10,00} &= L_{kj}^{00,01} = 0. \\ L_{jk}^{10,10} &= \frac{16T}{25p} \sum_{l=1}^{K} \frac{m_j}{m_k} \frac{X_j X_l}{(m_j + m_l)^2 \mathcal{D}_{jl}} \\ &\quad \times \left\{ \left( \delta_{kl} - \delta_{jk} \right) \left[ \frac{15}{2} m_k^2 + \frac{25}{4} m_l^2 - 3m_l^2 B_{jl}^* \right] \\ &\quad -4m_k m_l A_{jl}^* \left( \delta_{kl} + \delta_{jk} \right) \left[ 1 + \frac{5}{3\pi} \left( \frac{C_{j,\text{rot}}}{R\xi_{jl}} + \frac{C_{l,\text{rot}}}{R\xi_{lj}} \right) \right] \right\}. \end{split}$$

$$L_{jk}^{10,01} = \frac{32T}{5\pi p C_{k,\text{int}}} \sum_{l=1}^{K} \frac{m_k A_{kl}^*}{(m_k + m_l) \mathcal{D}_{kl}} \left(\delta_{jl} + \delta_{jk}\right) X_k X_l \frac{C_{k,\text{rot}}}{R\xi_{kl}}.$$

$$\begin{split} L_{jj}^{10,01} &= \frac{16}{3\pi} \frac{m_j X_j^2}{\mu_j C_{j,\text{int}}} \frac{C_{j,\text{rot}}}{R\xi_{jj}} \\ &+ \frac{32TR}{5\pi p C_{j,\text{int}}} \sum_{l \neq j}^K \frac{m_j A_{jl}^*}{(m_j + m_l) \mathcal{D}_{jl}} X_j X_l \frac{C_{j,\text{rot}}}{R\xi_{jl}}. \\ L_{jk}^{01,10} &= L_{kj}^{10,01}. \end{split}$$



#### <u>Alternative diffusion flux formulations</u>

- Mixture-average formulation

$$\vec{j}_{i} = -D_{i}^{M} p \frac{w_{i}}{x_{i}} \operatorname{grad} x_{i} - D_{i}^{T} \operatorname{grad} (\ln T) . \qquad (12.25)$$

$$D_{i}^{M} = \frac{1 - w_{i}}{\sum_{j \neq i} x_{j} / D_{ij}}$$

$$\underline{Binary \ diffusion \ coefficient} : \qquad (Lennard-Jones \ potentials \rightarrow Binary \ diffusion \ coefficients, see \ Lec \ 3)}$$



#### Alternative diffusion flux formulations

- Define non-dimensional Lewis number for each specie :  $Le_i =$ 

$$= \frac{\alpha}{\rho D_{im} c_p}.$$

λ

- Use this to rewrite your conservation equations :

$$\frac{\partial(\rho h)}{\partial t} + \nabla \cdot (\rho \boldsymbol{u} h) - \nabla \cdot \left(\frac{\lambda}{c_p} \nabla h\right) = \nabla \cdot \left(\frac{\lambda}{c_p} \sum_{i=1}^{N_{\rm s}} \left(\frac{1}{Le_i} - 1\right) h_i \nabla Y_i\right),$$
$$\frac{\partial(\rho Y_i)}{\partial t} + \nabla \cdot (\rho \boldsymbol{u} Y_i) - \nabla \cdot \left(\frac{\lambda}{Le_i c_p} \nabla Y_i\right) = \dot{\omega}_i, \qquad i = 1, \dots, N_{\rm s}.$$

- Lewis number can be then species-wise <u>constant</u>, <u>depend on temperature</u> or <u>Le = 1</u>
- Le =1 is justified if turbulent mixing dominates the flow



Driving force ⇒ ↓ Flux	Velocity gradient	Temperature gradient	Concentration gradient
Momentum	Newton's law $[\mu]$		
Energy		Fourier's law [λ]	Dufour effect $[D_i^T]$
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**Fig. 5.8.** Fluxes and driving forces in transport processes (Onsager 1931, Hirschfelder et al. 1964)





### Three levels of detail

- Flame sheet
  - Discontinuity
  - Conservation of mass and energy
  - Rankine-Hugoniot
  - Analytical analysis + combustion models
- <u>Reaction sheet level</u>
  - Includes the preheat zone
  - Finite thickness
  - Discontinuous reaction sheet
  - Analytical analysis + combustion models
- <u>Complete structure</u>
  - Fundamental understanding
  - Combustion models



- Solving the presented conservation equations is only possible via numerical methods
- Domain discretization for numerical methods should resolve the flame structure
- Numerical solution very CPU-intense  $\rightarrow$  modeling above 1D is mandatory
- Often the machine-crunched brute-force solution does not help us to understand the flame
- Simplified analytical analysis may give a lot of insight



- Zeldovich and Kamenetskii developed the following analytical approach:
  - Assume 1D reacting flow
  - Assume steady-state flame (equivalent to steady flame front in bunsen burners)
  - One-step global reaction : Fuel (F)  $\rightarrow$  Products (P)
  - Global reaction rate  $r = -\rho w_F k = -\rho w_F \cdot A \cdot \exp(-E/RT)$
  - Multicomponent diffusion negligible and diffusion does not depend on location
  - The above results the following equations :

$$D\frac{\partial^2 w_{\rm F}}{\partial z^2} - v\frac{\partial w_{\rm F}}{\partial z} - w_{\rm F} \cdot A \cdot \exp\left(-\frac{E}{RT}\right) = 0 \qquad (8.1)$$

$$\frac{\lambda}{\rho c_p} \frac{\partial^2 T}{\partial z^2} - \upsilon \frac{\partial T}{\partial z} + w_F \frac{h_P - h_F}{c_p} \cdot A \cdot \exp\left(-\frac{E}{RT}\right) = 0 \quad (8.2)$$



- Furthermore assuming Le = 1 and the two equations become similar
- Change of variables and integration on both sides results single equation

$$\alpha \frac{\mathrm{d}^2 \delta}{\mathrm{d}z^2} - \upsilon \frac{\mathrm{d}\delta}{\mathrm{d}z} - \delta \cdot A \cdot \exp\left[-\frac{E}{R(T_{\mathrm{b}} - \delta)}\right] = 0$$

$$\delta = T_{\rm b} - T = [(h_{\rm P} - h_{\rm F})/c_p] w_{\rm F}$$

- We want to solve the profiles of temperature and species and particular velocity
- (steady state) + 2<sup>nd</sup> order ODE  $\rightarrow$  eigenvalue problem  $\rightarrow$  eigenvalue  $v_{\rm L} = \sqrt{\frac{\alpha}{\tau}}$
- Mass burning flux (lam. Flame speed) is the fundamental property of the flame

- Implications of Zeldovich's analysis  $v_{\rm L} = \sqrt{\frac{\alpha}{\tau}}$   $\tau = 1/k = [A \cdot \exp(-E/RT)]^{-1}$ 
  - Laminar flame speed depends on the diffusivity  $\alpha$  and characteristic time of reaction au
  - Flame propagation is caused by diffusive processes
  - Necessary gradients (wave) are sustained by the chemical reaction
- Furthermore, one can show that the flame thickness follows :  $\ell_F = \frac{\alpha}{v_L}$



• How does the mixture temperature effect?



• Mixture temperature effect : Stoichiometric CH4-air flame



Upstream Temperature of Mixture (K)

C.K. Law Combustion Physics : Fig. 7.7.12

• How does the ambient pressure effect?



• Pressure effect on CH4-air flame : Laminar flame speed (a) and burning flux (b)





• Pressure effect on flame thickness



C.K. Law Combustion Physics : Fig. 7.7.11



• How about the fuel ?



- Adiabatic flame temperature and Lewis number effect
  - Higher heat of combustion  $\rightarrow$  faster propagation



C.K. Law Combustion Physics : Fig. 7.7.2



- Adiabatic flame temperature of the fuel dominates the flame speed
- Butane Heptane share also very similar adiabatic flame temperature curves



C.K. Law Combustion Physics : Fig. 7.7.4



• However, molecular structure has a role, here fuels diluted to share same T<sub>ad</sub> at stoich.



C.K. Law Combustion Physics : Fig. 7.7.6





C.K. Law Combustion Physics : Fig. 7.7.13



### **1D flame calculations in practice**

- Cantera has full capabilities to solve detailed chemistry in 1D
  - http://www.cantera.org/docs/sphinx/html/cython/examples.html#one-dimensional-flames
  - In particular we are interested in freely propagating adiabatic premixed flame : http://www.cantera.org/docs/sphinx/html/cython/examples/onedim\_adiabatic\_flame.html
  - Note the choice of the mixture-averaged or multicomponent diffusion model
  - Cantera will discretize the 1D domain into N solution points
  - Resolution and time steps are controlled by the solver in run-time
  - However at high temperatures and pressures user has to modify the grid resolution criteria, time step and solver tolerances setup.

